



AMCA International

Minimizing System Effect

Mark Bublitz

Vice President - Engineering

mbublitz@nyb.com

David Maletich

Director - Marketing

dmaletich@nyb.com



The New York Blower Company

Air System Engineering & Technology (ASET) Conference-US

San Antonio, TX • Hyatt Regency San Antonio Riverwalk • March 6 - 7, 2018

Copyright
AMCA International • www.amca.org

Professional Development Hours (PDH) Certificates

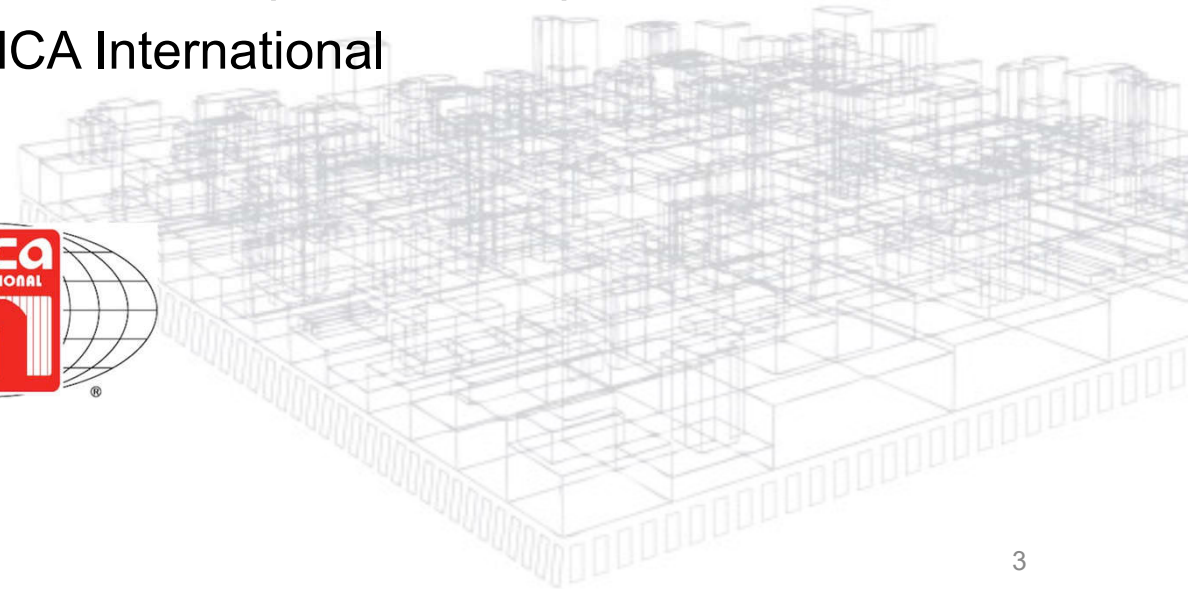
The Air Movement and Control Association International (AMCA), has met the standards and requirements of the Registered Continuing Education Providers Program. Credit earned on completion of this program will be reported to the RCEP. A certificate of completion will be issued to each participant. As such, it does not include content that may be deemed or construed to be an approval or endorsement by NCEES or RCEP.



Copyright Materials

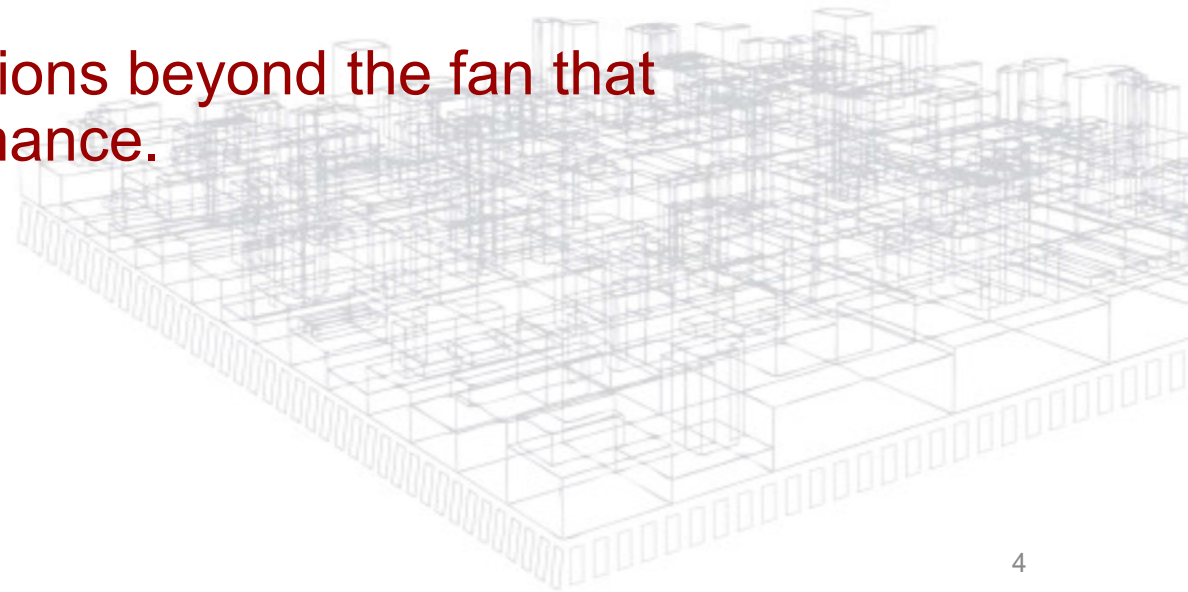
This educational activity is protected by U.S. and International copyright laws. Reproduction, distribution, display, and use of the educational activity without written permission of the presenter is prohibited.

© AMCA International



Learning Objectives

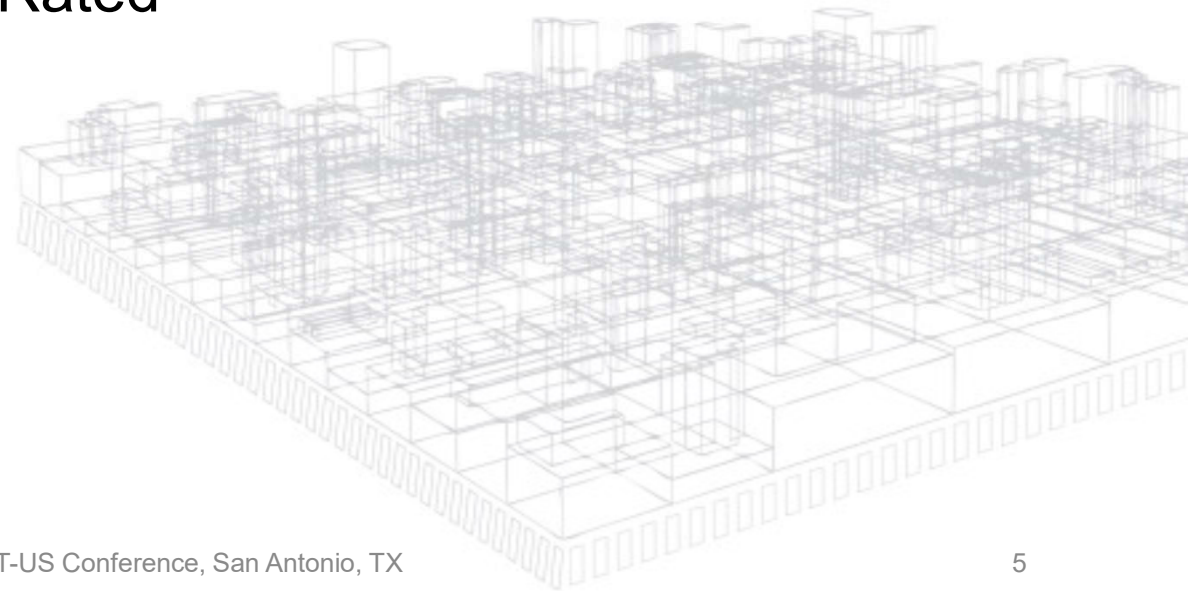
- Review the primary engineering principles involved in fan operation, rating, and performance.
- Understand conditions beyond the fan that impact fan performance.



Fan Fundamentals

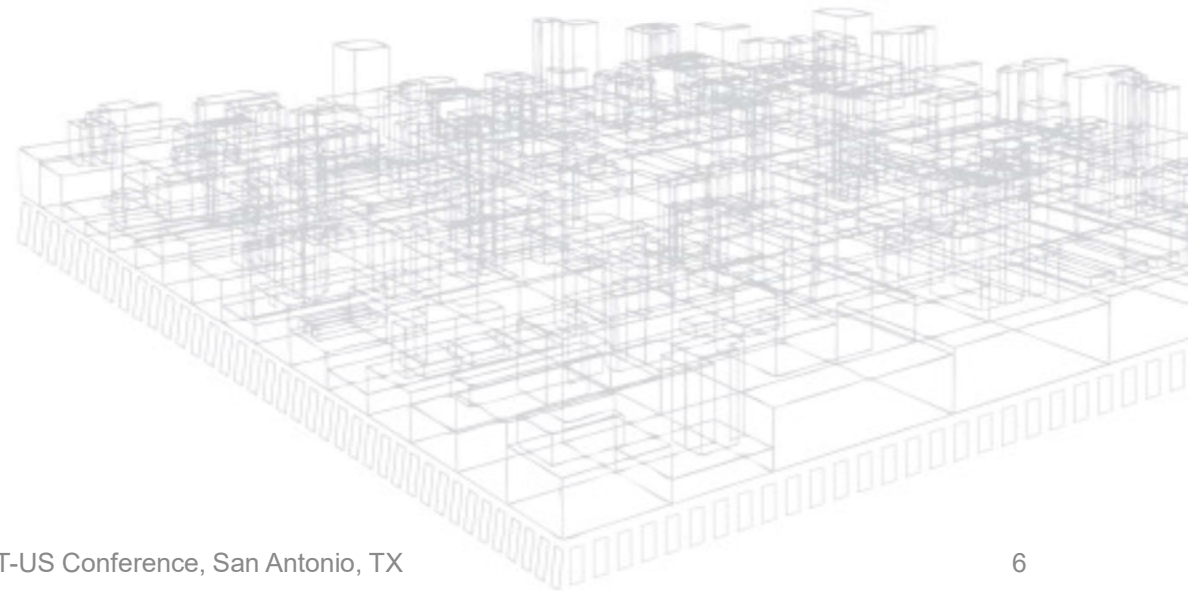
Subjects

- Fan Laws
- How Fans are Tested and Rated
- Various Fan Types
- System Effects

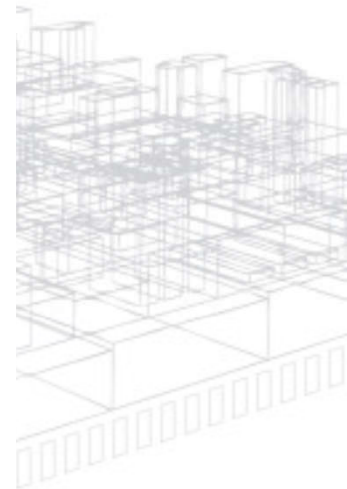
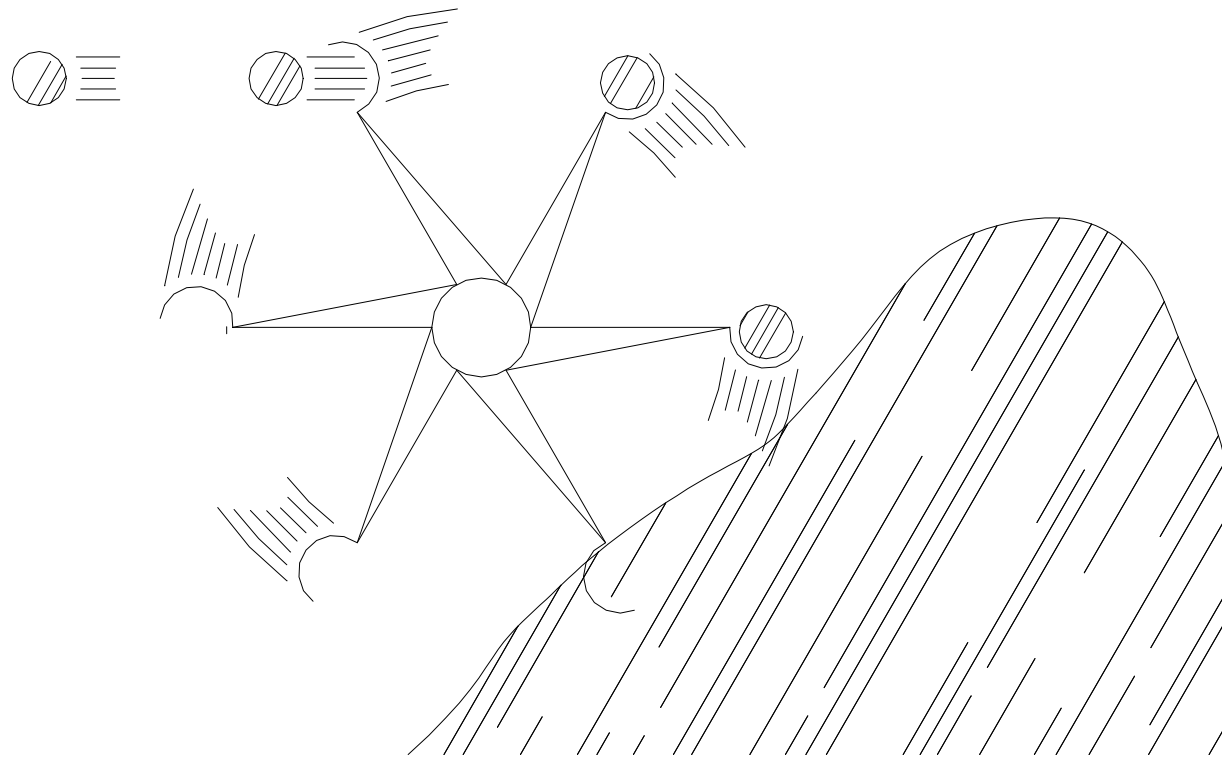


Key Terms

- CFM
- SP (Ps)
- VP (Pv)
- RPM
- BHP
- kW

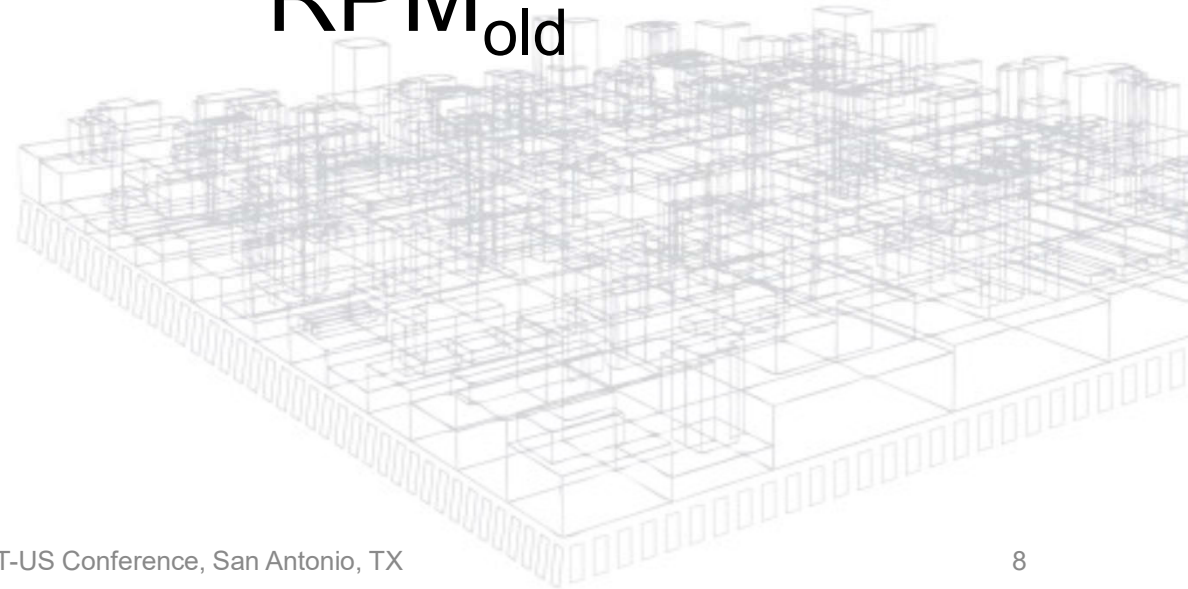


Fan Laws



Fan Laws

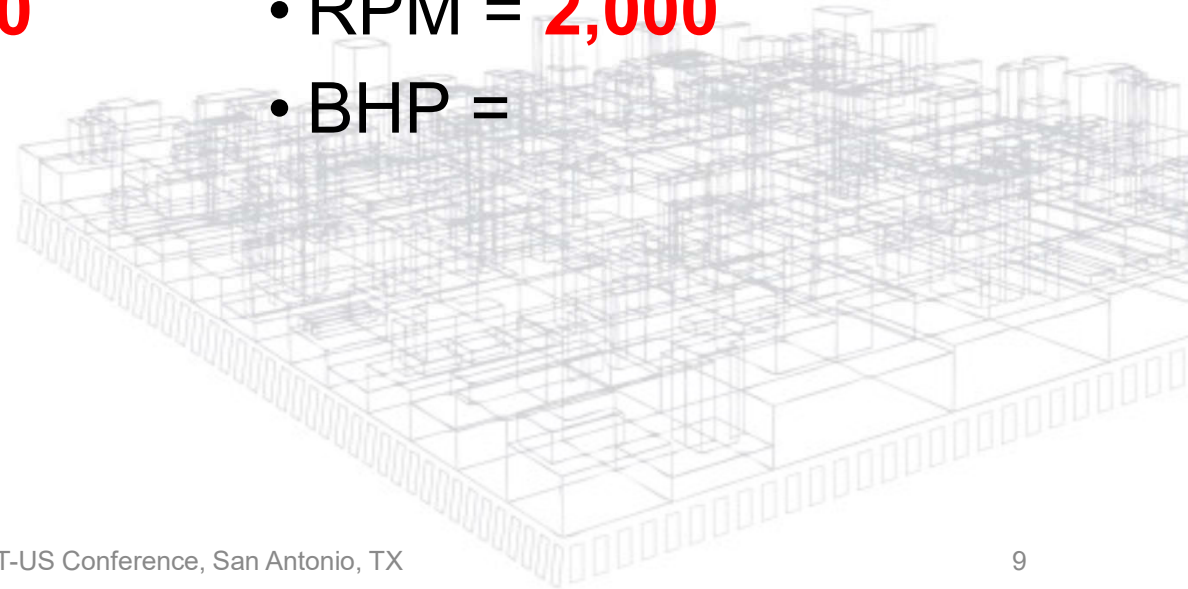
$$\frac{\text{CFM}_{\text{new}}}{\text{CFM}_{\text{old}}} = \frac{\text{RPM}_{\text{new}}}{\text{RPM}_{\text{old}}}$$



Fan Laws

- CFM = **10,000**
- SP = 1"
- RPM = **1,000**
- BHP = 10

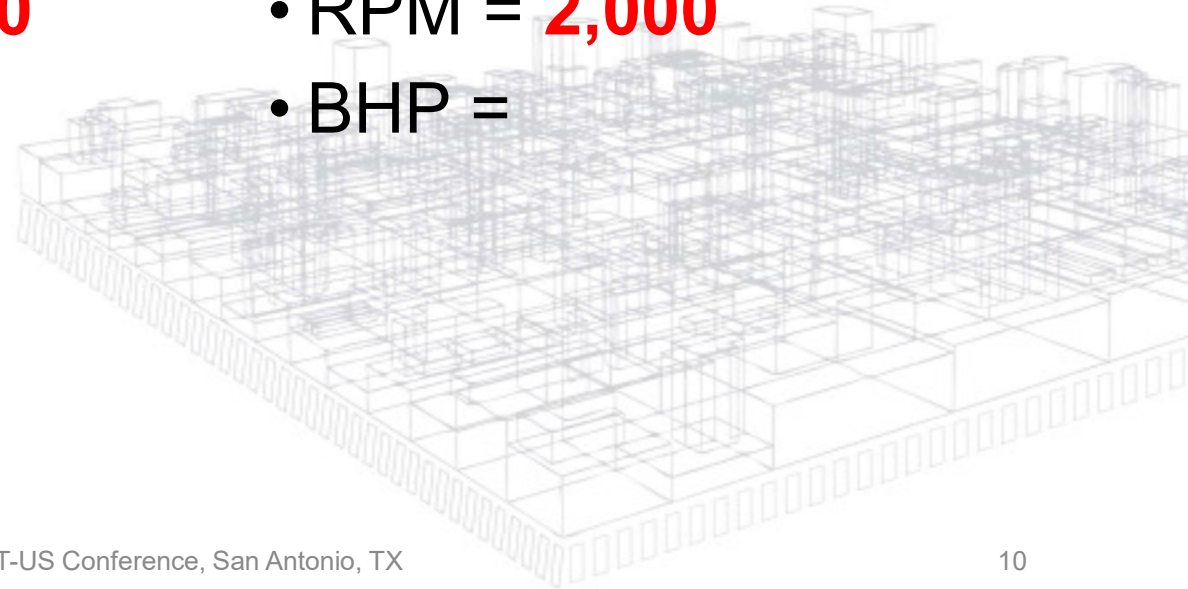
- CFM =
- SP =
- RPM = **2,000**
- BHP =



Fan Laws

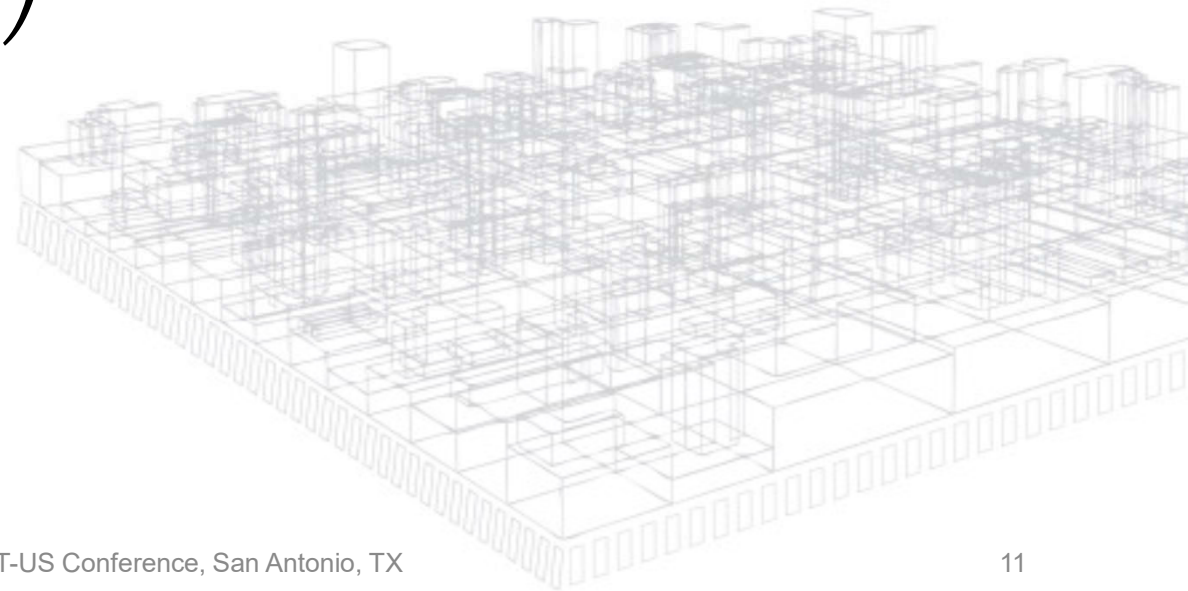
- CFM = **10,000**
- SP = 1"
- RPM = **1,000**
- BHP = 10

- CFM = **20,000**
- SP =
- RPM = **2,000**
- BHP =



Fan Laws

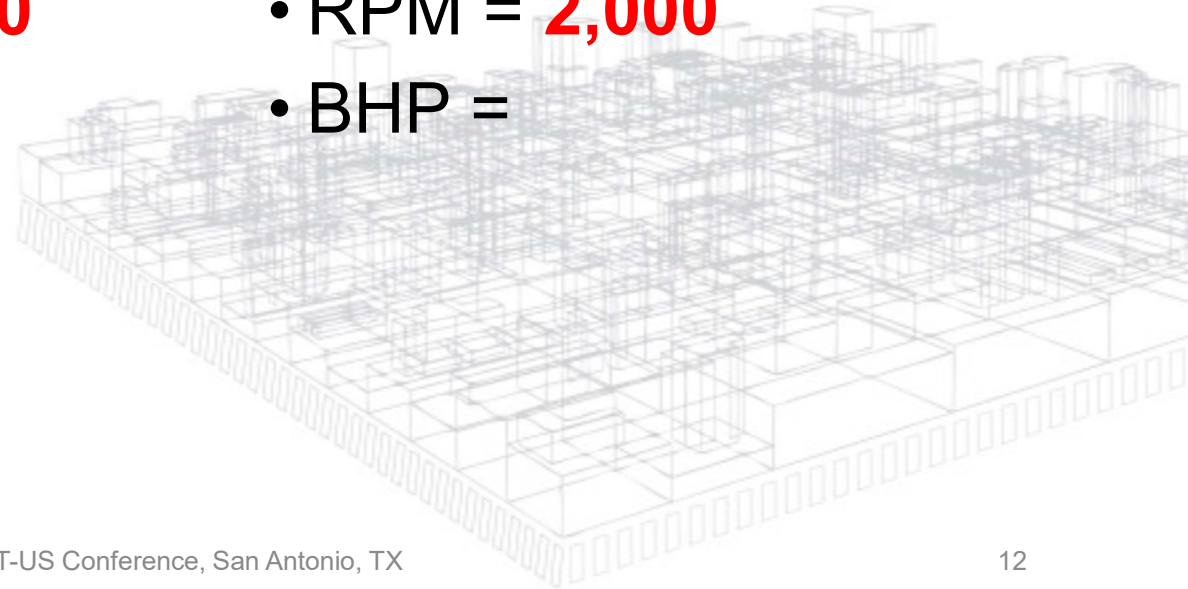
$$\frac{SP_{new}}{SP_{old}} = \left(\frac{RPM_{new}}{RPM_{old}} \right)^2$$



Fan Laws

- CFM = 10,000
- SP = **1"**
- RPM = **1,000**
- BHP = 10

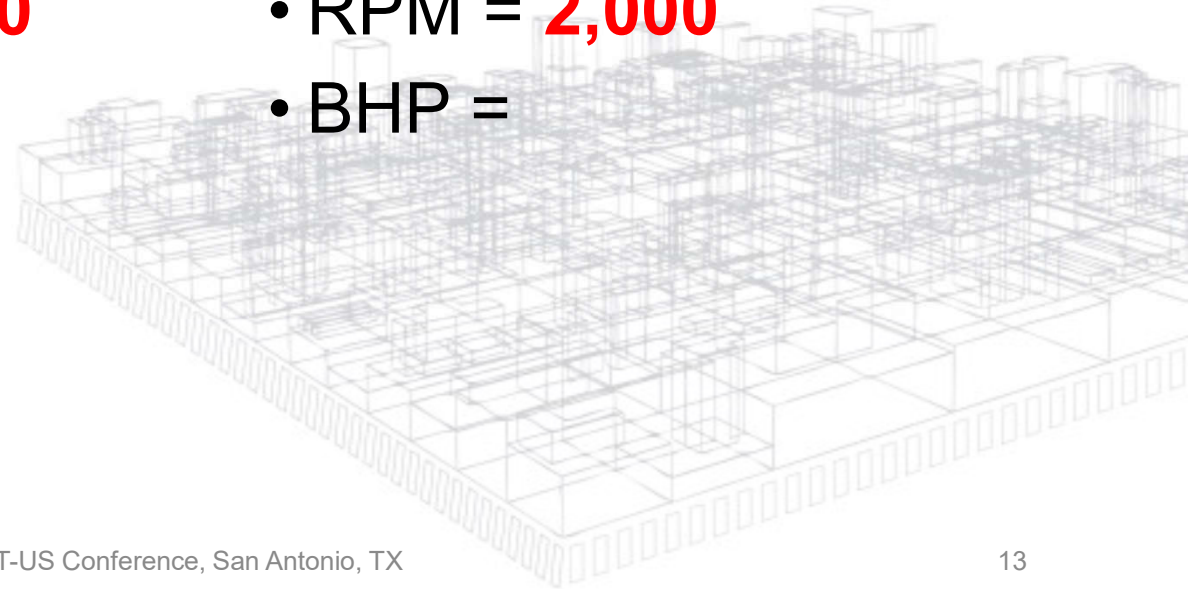
- CFM = 20,000
- SP =
- RPM = **2,000**
- BHP =



Fan Laws

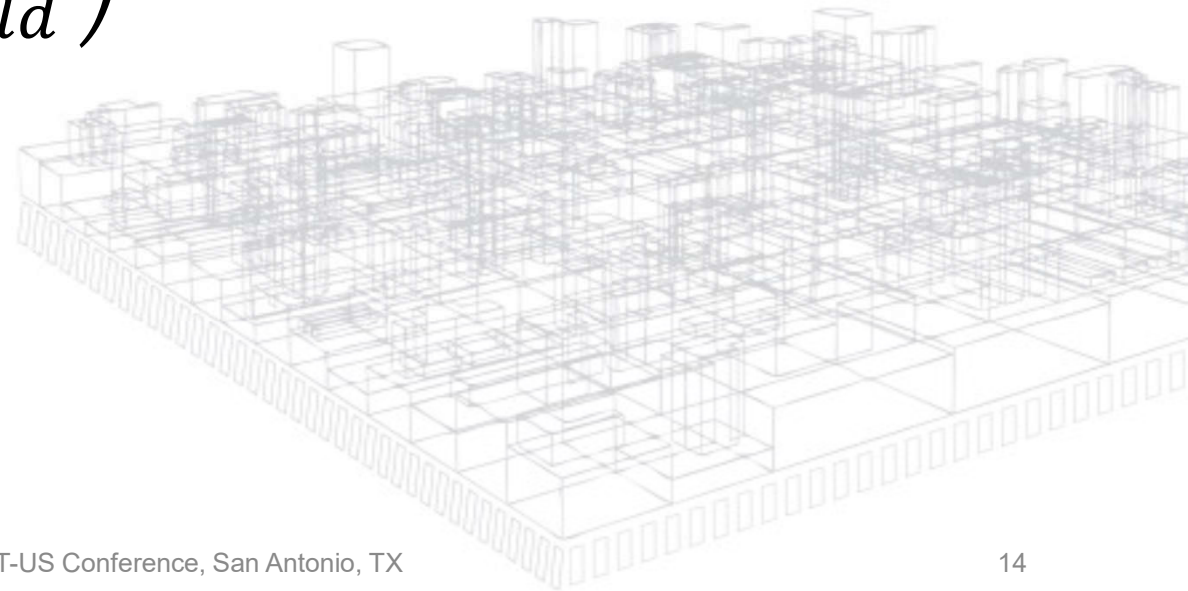
- CFM = 10,000
- SP = **1"**
- RPM = **1,000**
- BHP = 10

- CFM = 20,000
- SP = **4"**
- RPM = **2,000**
- BHP =



Fan Laws

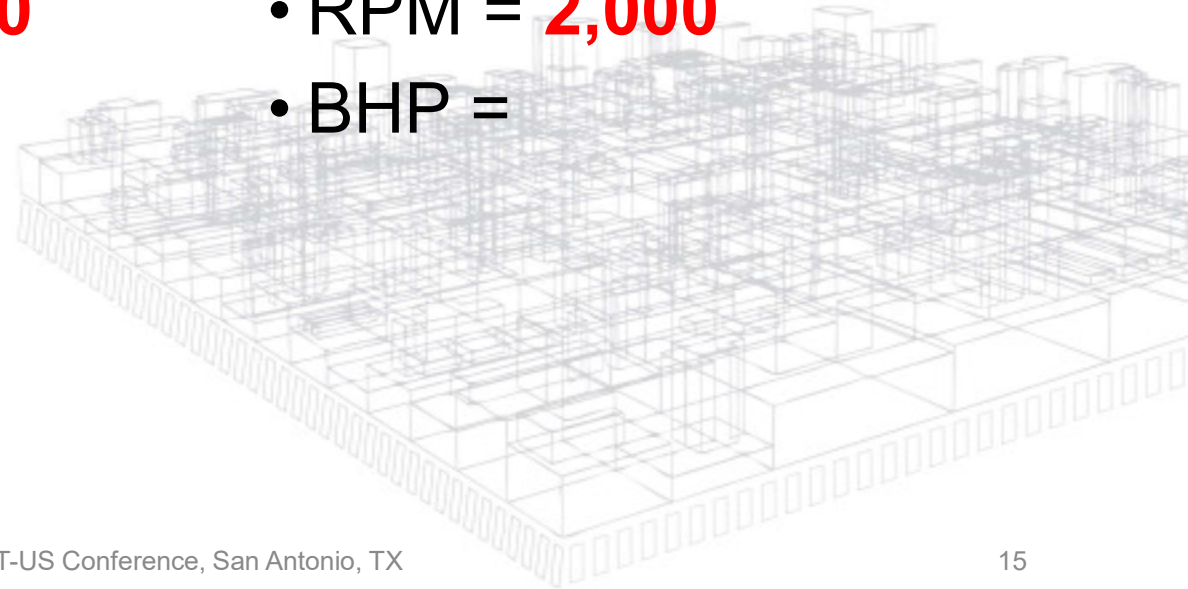
$$\frac{BHP_{new}}{BHP_{old}} = \left(\frac{RPM_{new}}{RPM_{old}} \right)^3$$



Fan Laws

- CFM = 10,000
- SP = 1"
- RPM = **1,000**
- BHP = **10**

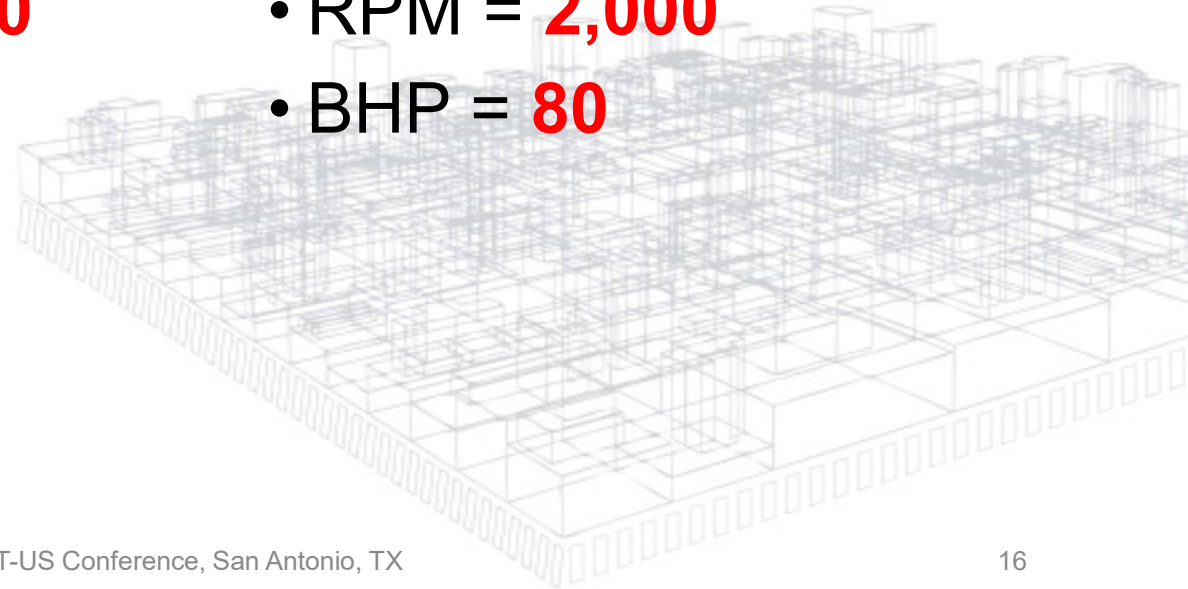
- CFM = 20,000
- SP = 4"
- RPM = **2,000**
- BHP =



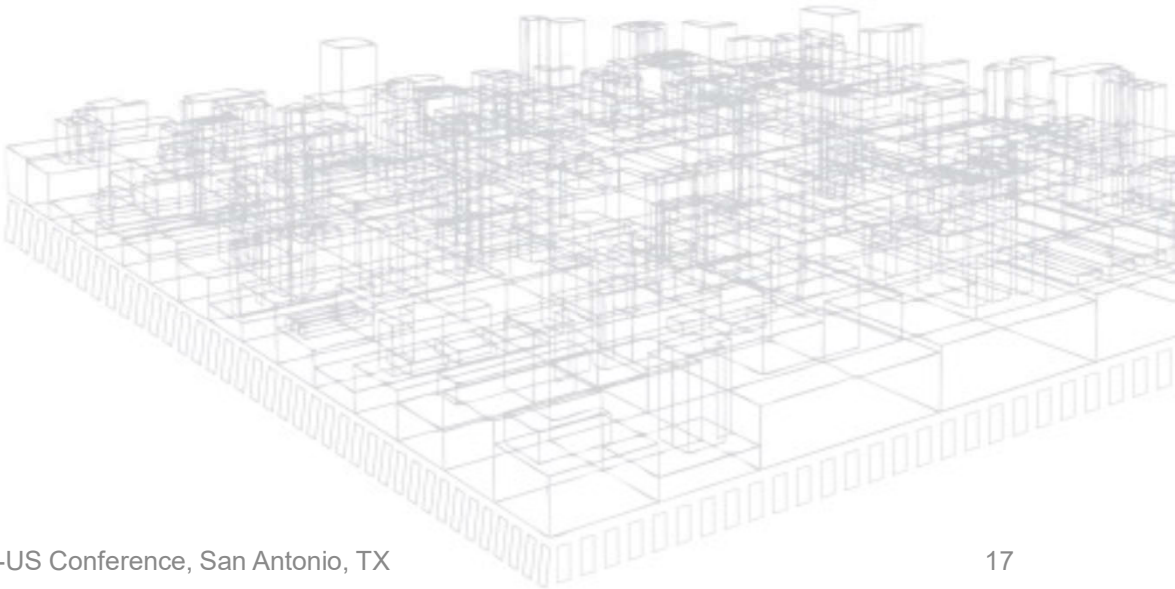
Fan Laws

- CFM = 10,000
- SP = 1
- RPM = **1,000**
- BHP = **10**

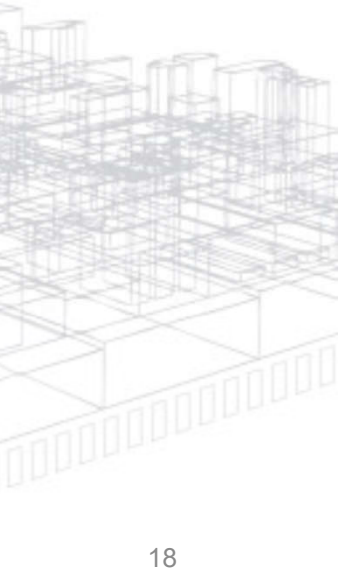
- CFM = 20,000
- SP = 4"
- RPM = **2,000**
- BHP = **80**



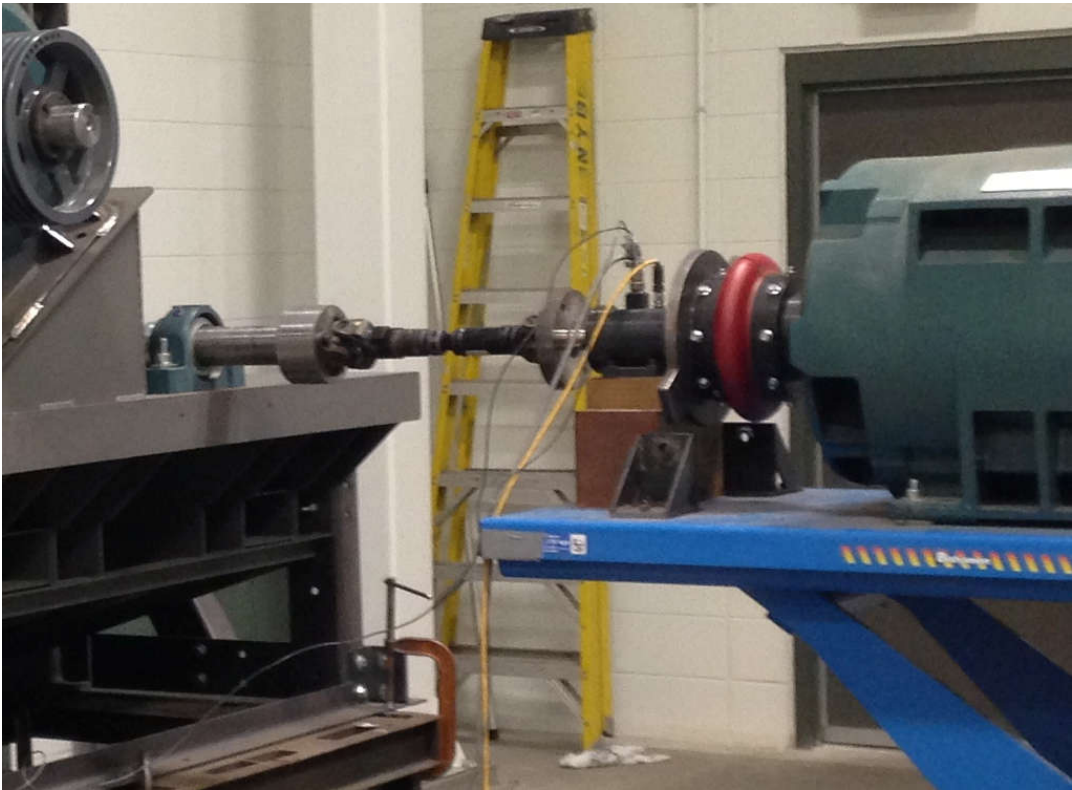
Testing



Testing



Testing



Testing



March 6–7 2018 www.aset-us.com

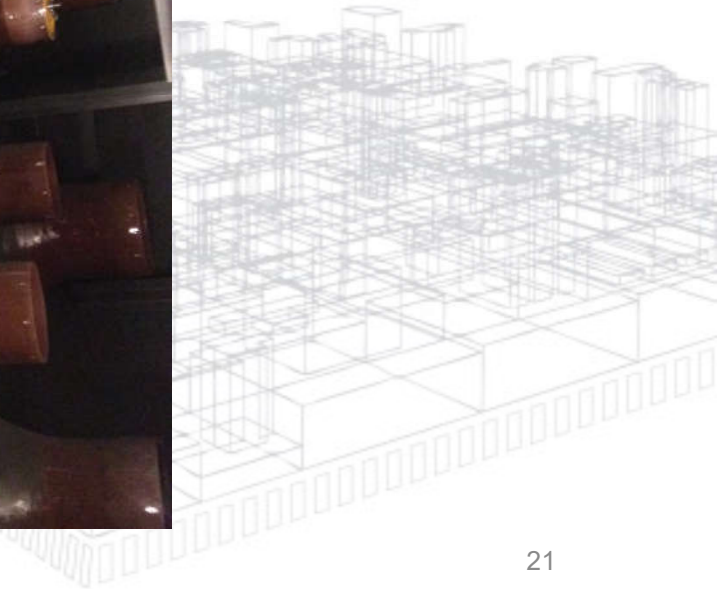
AMCA ASET-US Conference, San Antonio, TX

Testing

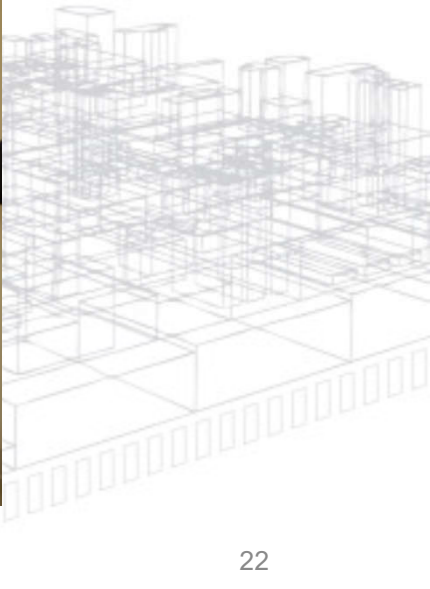


March 6–7 2018 www.aset-us.com

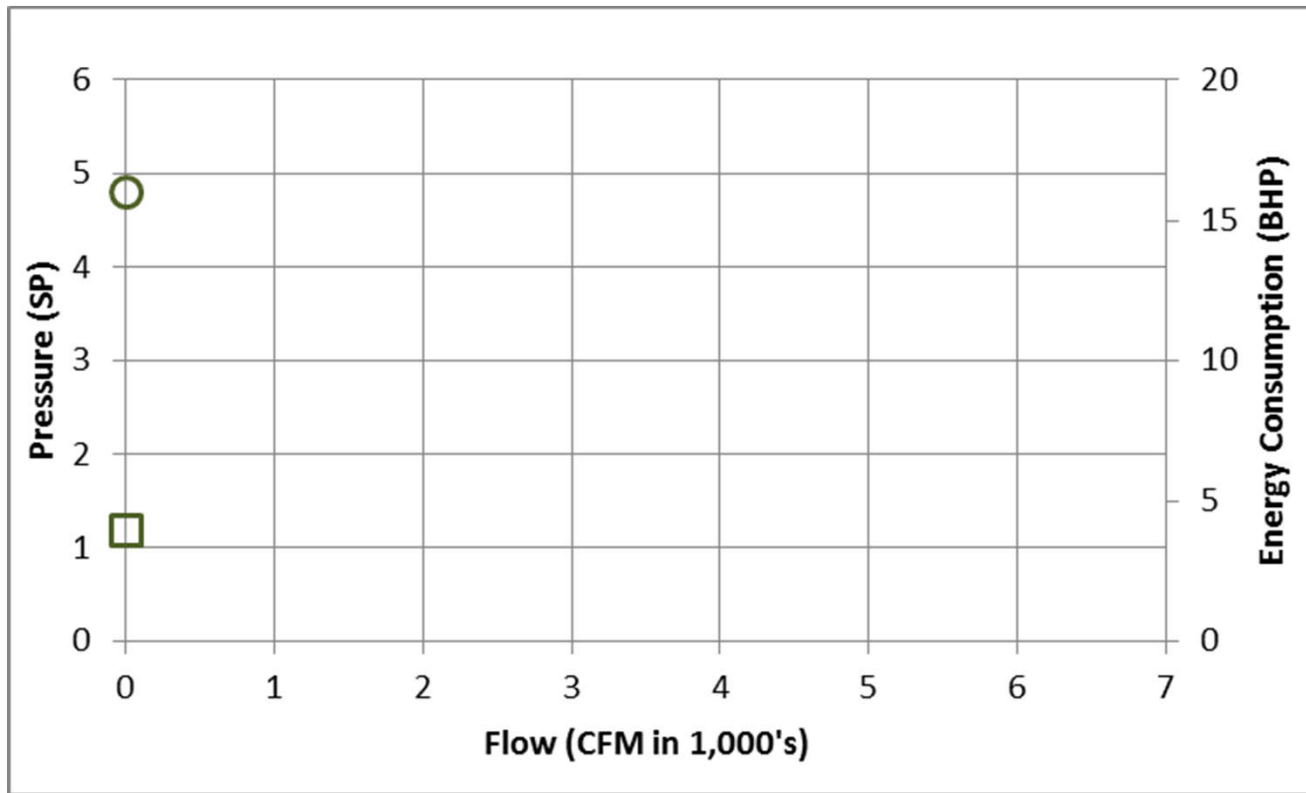
AMCA ASET-US Conference, San Antonio, TX



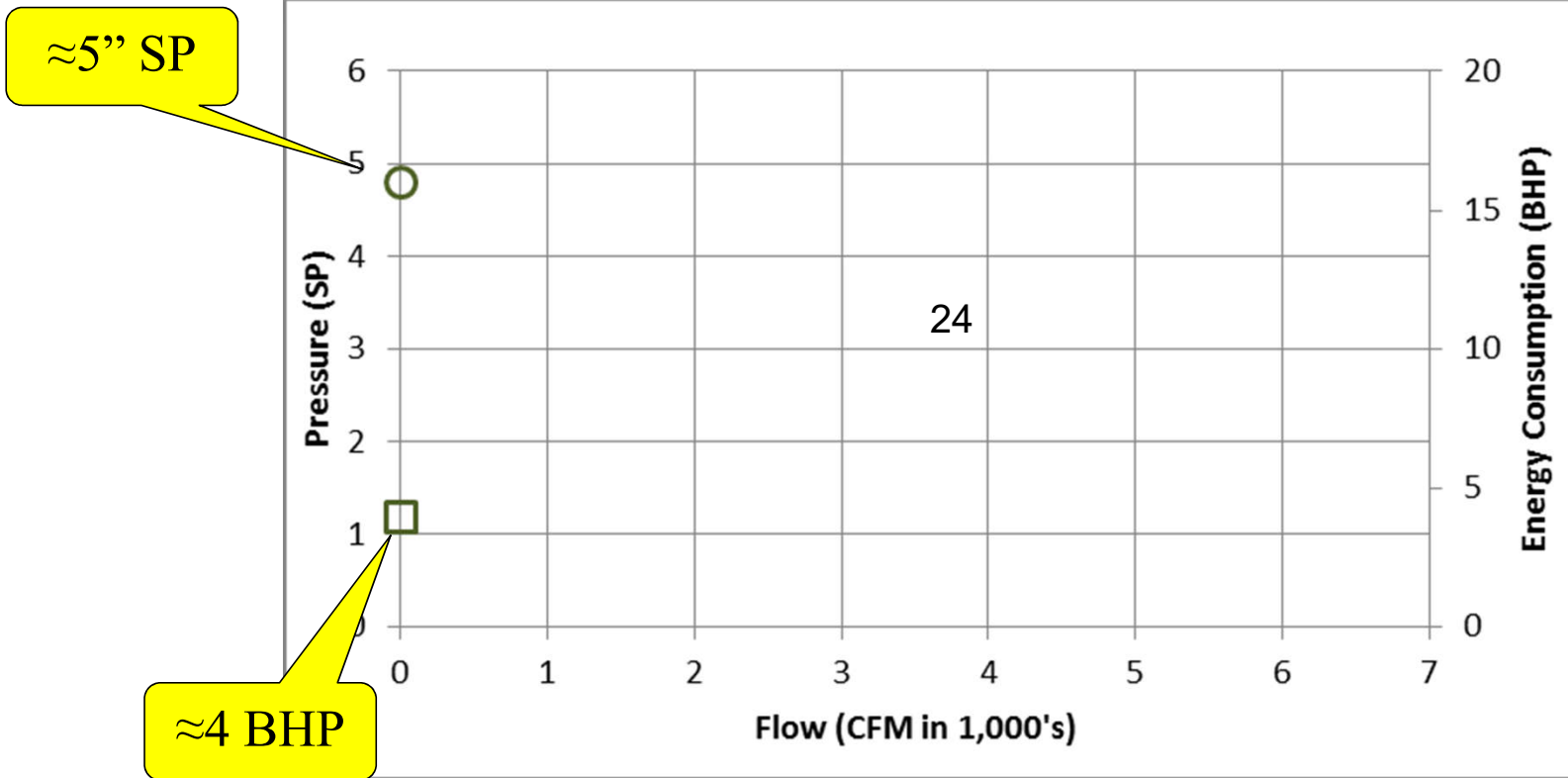
Testing



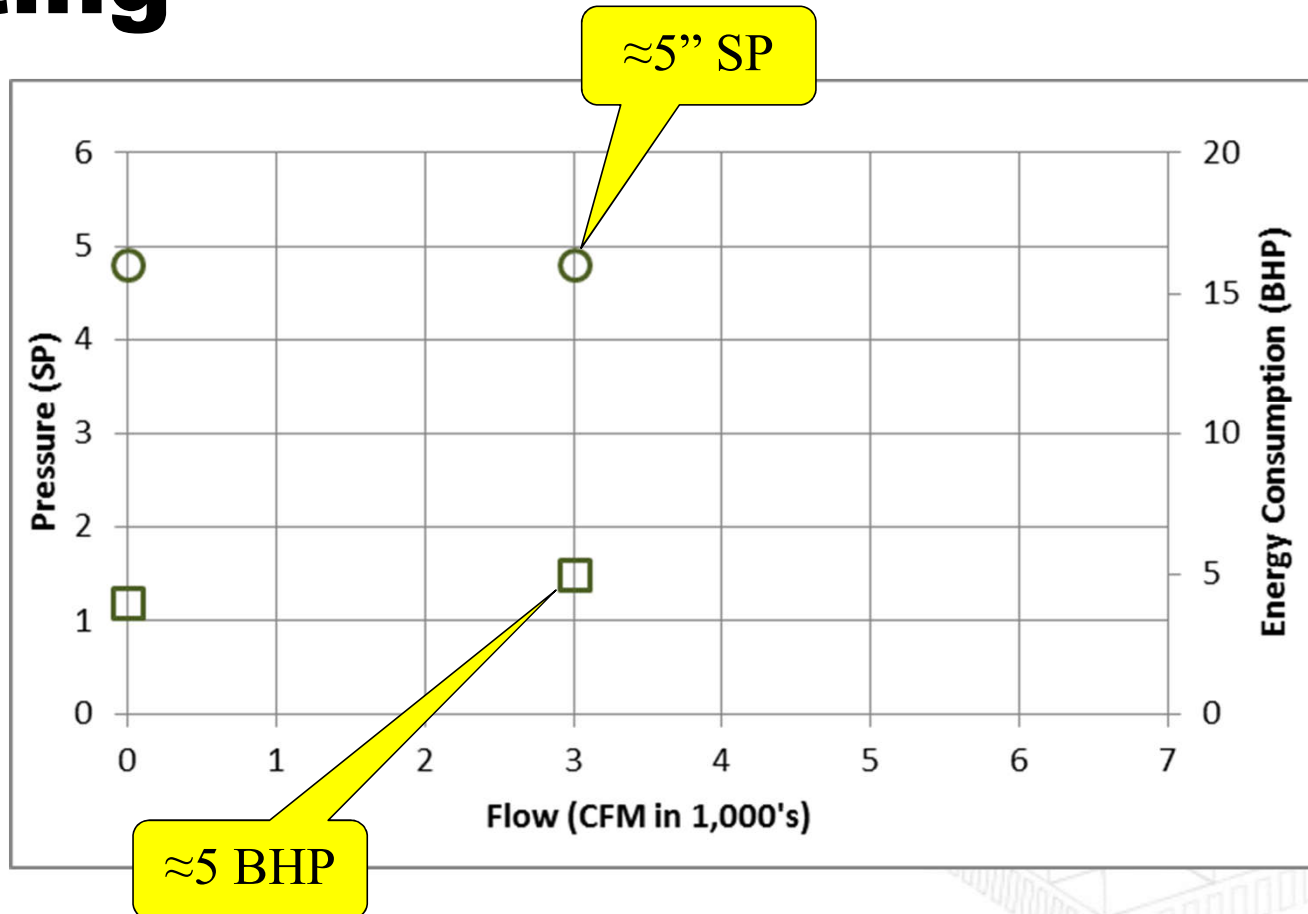
Testing



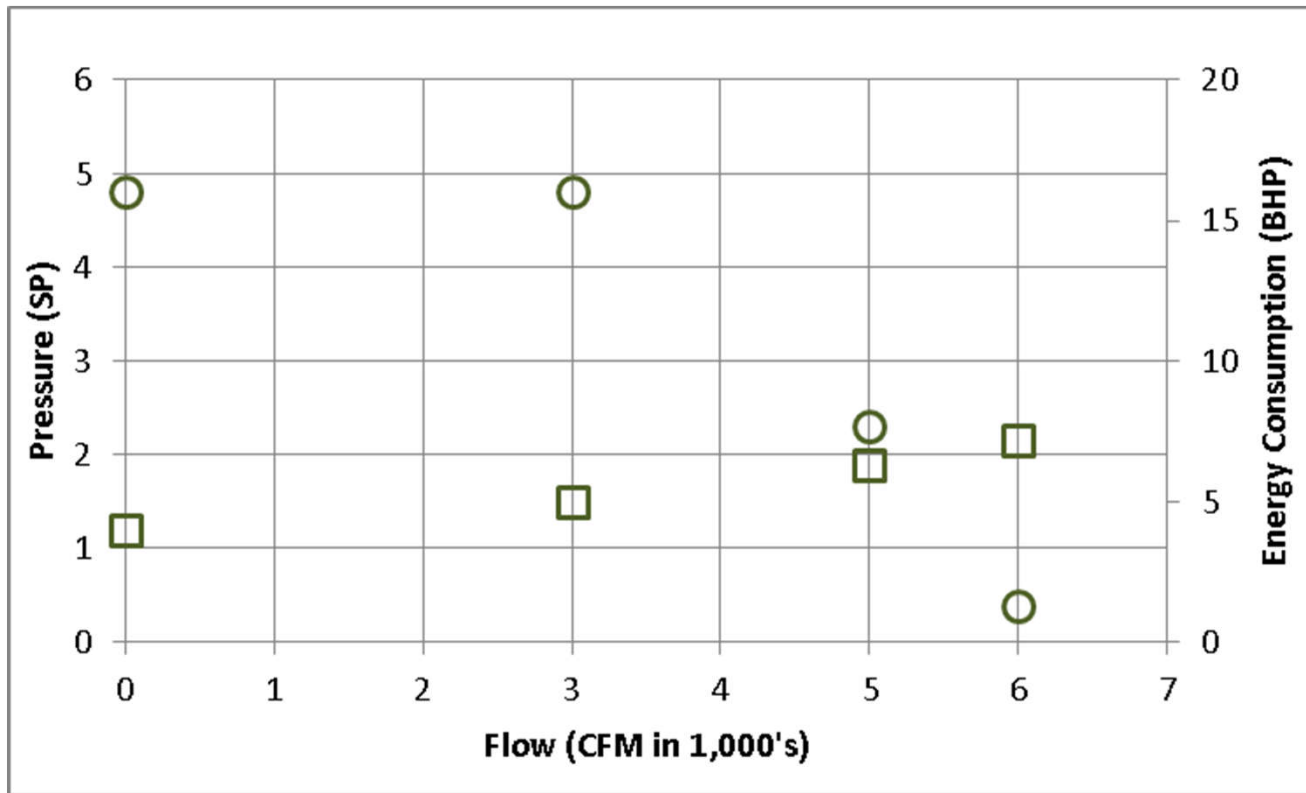
Testing



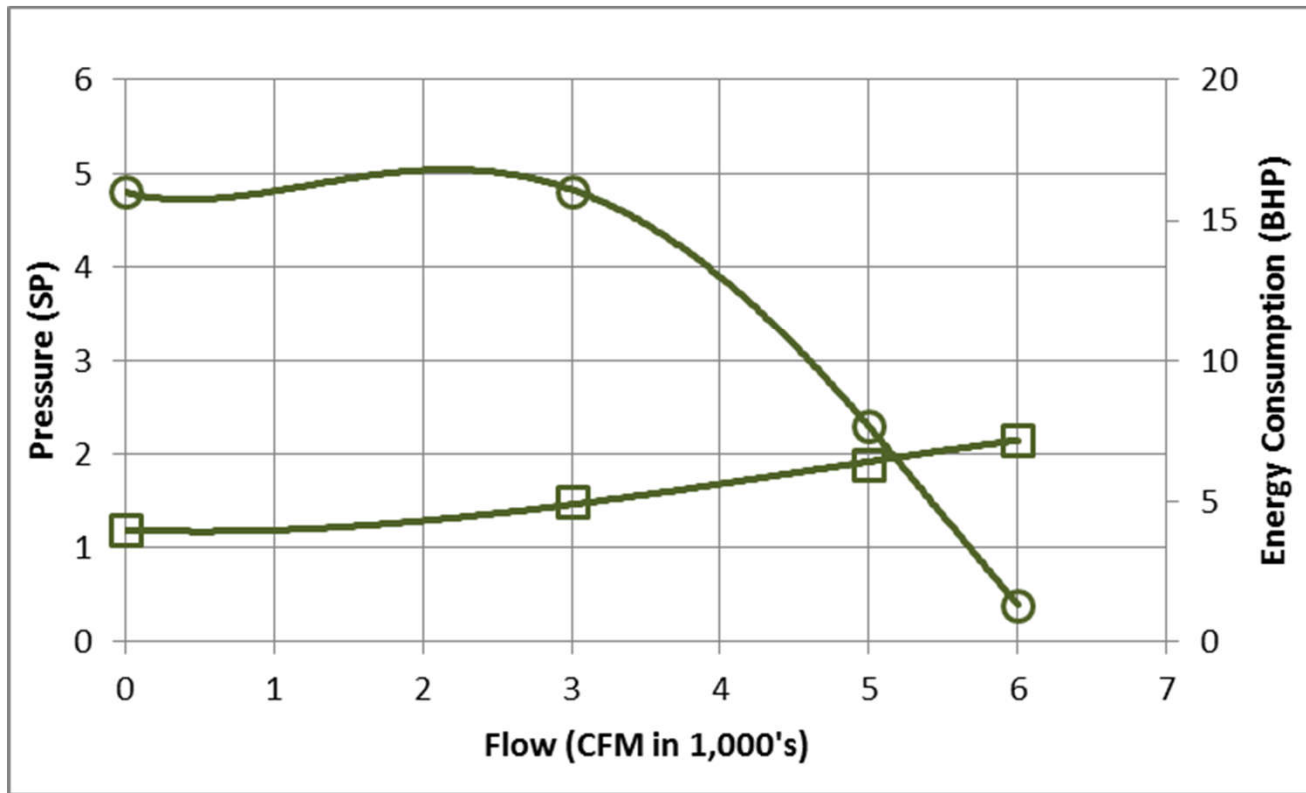
Testing



Testing



Fan Performance Curve



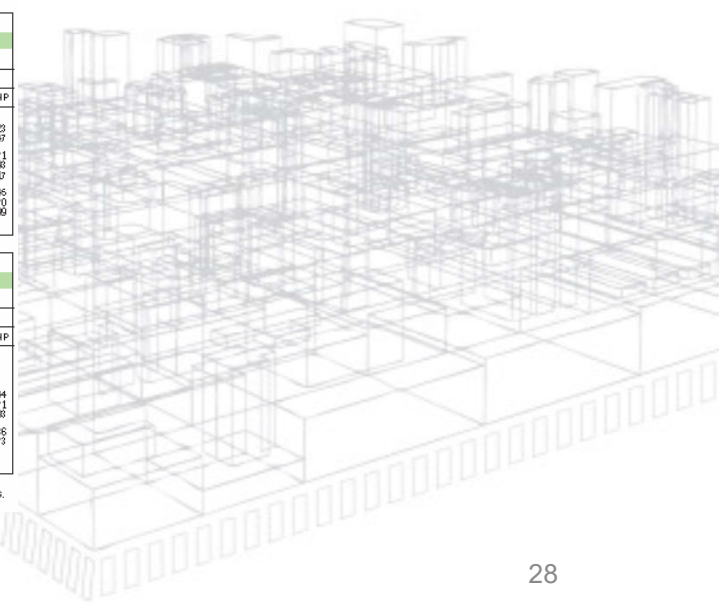
Catalog Performance Tables

MODEL 405		WITH EVASE		Wheel diameter: 40.3" Wheel circumference: 10.5'		Outlet area: 10.2 sq. ft. Maximum BHP = 24.5 $\left(\frac{\text{RPM}^3}{1000}\right)$		AF-30=2035 RPM AF-40=2470 RPM AF-50=2640 RPM				
CFM	OV	18"SP	22"SP	26"SP	28"SP	30"SP	34"SP	36"SP	42"SP	46"SP	48"SP	50"SP
		RPM BHP	RPM BHP	RPM BHP	RPM BHP	RPM BHP	RPM BHP	RPM BHP	RPM BHP	RPM BHP	RPM BHP	RPM BHP
2400	2349	1940 76.9	1891 96	1828 116	1896 125	1860 136	2095 157	2200 189	2210 204	2422 263	2478 284	
2600	2500	1874 85.9	1836 106	1766 126	1832 135	1806 146						
2800	2657	1816 95.5	1775 115	1698 136	1768 146	1742 157						
3000	3112	1866 107	1775 126	1686 146	1747 157	1721 168	2000 170	2111 194	2221 218	2320 242	2429 268	2478 281
3400	3335	1721 119	1657 140	1528 162	1580 173	1554 184	2005 186	2127 209	2241 235	2339 260	2438 286	2492 310
3700	3520	1776 132	1690 154	1577 177	1632 188	1607 199	2010 201	2133 227	2249 251	2349 279	2448 304	2503 318
3800	3526	1866 147	1799 170	1674 192	1728 203	1703 214	2018 220	2142 244	2257 271	2356 298	2455 324	2510 318
4200	4129	1820 164	1704 188	1589 212	1643 223	1618 234	2026 236	2151 260	2267 287	2366 314	2465 340	2520 318
4400	4398	1891 181	1771 206	1656 232	1710 243	1685 254	2034 251	2159 275	2275 302	2374 329	2473 355	2528 318
4700	4638	2085 202	1944 228	1829 254	1883 265	1858 276	2046 276	2171 300	2287 327	2386 354	2485 380	2540 318

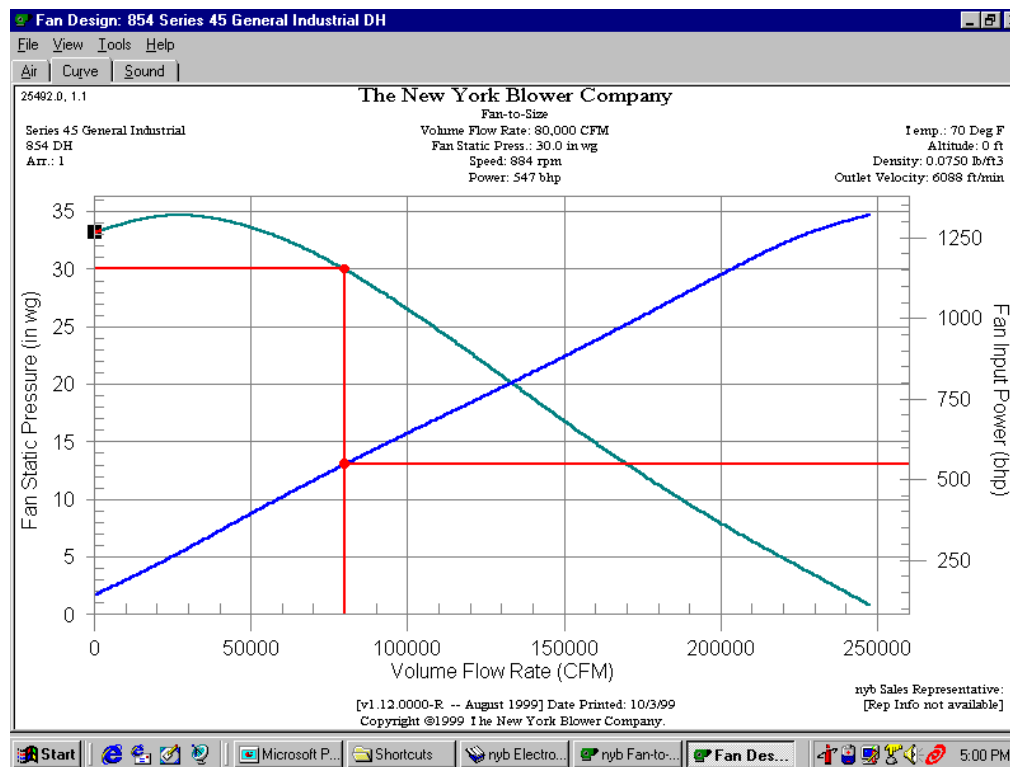
MODEL 445		WITH EVASE		Wheel diameter: 44.5" Wheel circumference: 11.7'		Outlet area: 12.5 sq. ft. Maximum BHP = 40.3 $\left(\frac{\text{RPM}^3}{1000}\right)$		AF-30=1890 RPM AF-40=2230 RPM AF-50=2400 RPM				
CFM	OV	18"SP	22"SP	26"SP	28"SP	30"SP	34"SP	36"SP	42"SP	46"SP	48"SP	50"SP
		RPM BHP	RPM BHP	RPM BHP	RPM BHP	RPM BHP	RPM BHP	RPM BHP	RPM BHP	RPM BHP	RPM BHP	RPM BHP
3000	2939	1409 59.6	1377 120	1351 144	1422 157	1380 169	1586 199.0	1606 226	2099 258	2194 294	2240 311	2288 323
3300	3264	1426 108	1364 131	1301 155	1370 170	1328 182	1527 203	1546 230	2036 266	2130 302	2176 319	2224 332
3600	3510	1466 118	1394 144	1322 162	1392 176	1350 188	1549 207	1568 234	2058 270	2152 306	2198 323	2246 337
3900	3766	1509 131	1436 158	1364 176	1434 190	1392 202	1594 224	1613 251	2103 287	2197 322	2243 339	2291 351
4200	3921	1556 146	1466 172	1392 190	1462 204	1420 216	1604 228	1623 255	2113 291	2207 326	2253 343	2301 353
4500	4077	1594 159	1497 187	1424 204	1494 218	1452 230	1614 242	1633 269	2123 295	2217 330	2263 349	2311 355
4800	4333	1661 175	1548 206	1474 226	1544 240	1502 252	1624 254	1643 281	2133 309	2227 344	2273 361	2321 361
5100	4589	1718 192	1601 224	1526 246	1596 260	1554 272	1636 264	1655 291	2143 323	2237 358	2283 367	2331 367
5400	4845	1780 213	1666 244	1590 266	1660 280	1618 292	1640 284	1659 311	2153 337	2247 372	2293 376	2341 376
5700	5101	1844 235	1735 264	1650 286	1720 300	1678 312	1660 304	1679 331	2163 351	2257 386	2303 390	2351 390

MODEL 495		WITH EVASE		Wheel diameter: 49.0" Wheel circumference: 12.3'		Outlet area: 15.3 sq. ft. Maximum BHP = 65.5 $\left(\frac{\text{RPM}^3}{1000}\right)$		AF-30=1675 RPM AF-40=2025 RPM AF-50=2170 RPM				
CFM	OV	18"SP	22"SP	26"SP	28"SP	30"SP	34"SP	36"SP	42"SP	46"SP	48"SP	50"SP
		RPM BHP	RPM BHP	RPM BHP	RPM BHP	RPM BHP	RPM BHP	RPM BHP	RPM BHP	RPM BHP	RPM BHP	RPM BHP
3600	2961	1267 116	1380 142	1300 162	1366 177	1324 189	1510 202	1529 229	1810 266	1905 302	1951 319	1997 326
4000	3262	1296 130	1404 156	1324 176	1390 191	1348 203	1526 212	1545 239	1826 286	1921 332	1967 349	2013 356
4400	3563	1326 144	1434 170	1354 190	1420 205	1378 217	1556 226	1575 253	1856 300	1951 346	1997 363	2043 370
4800	3864	1356 158	1464 194	1384 210	1450 225	1408 237	1586 246	1605 273	1886 320	1981 366	2027 383	2073 390
5200	4165	1386 172	1494 218	1414 238	1480 253	1438 265	1616 274	1635 301	1916 344	2011 390	2057 407	2103 414
5600	4466	1416 186	1524 242	1444 262	1510 277	1468 289	1646 298	1665 325	1946 368	2041 414	2087 431	2133 438
6000	4767	1446 200	1554 266	1474 286	1540 301	1500 313	1678 322	1697 349	1978 392	2073 438	2119 455	2165 462
6400	5068	1476 214	1584 290	1504 310	1570 325	1530 337	1708 336	1727 363	2008 416	2103 462	2149 479	2195 486
6800	5369	1506 228	1614 314	1544 334	1610 349	1570 361	1748 350	1767 377	2048 430	2143 476	2189 493	2235 500
7200	5670	1536 242	1644 338	1574 358	1640 373	1600 385	1778 369	1797 396	2078 444	2173 490	2219 507	2265 514

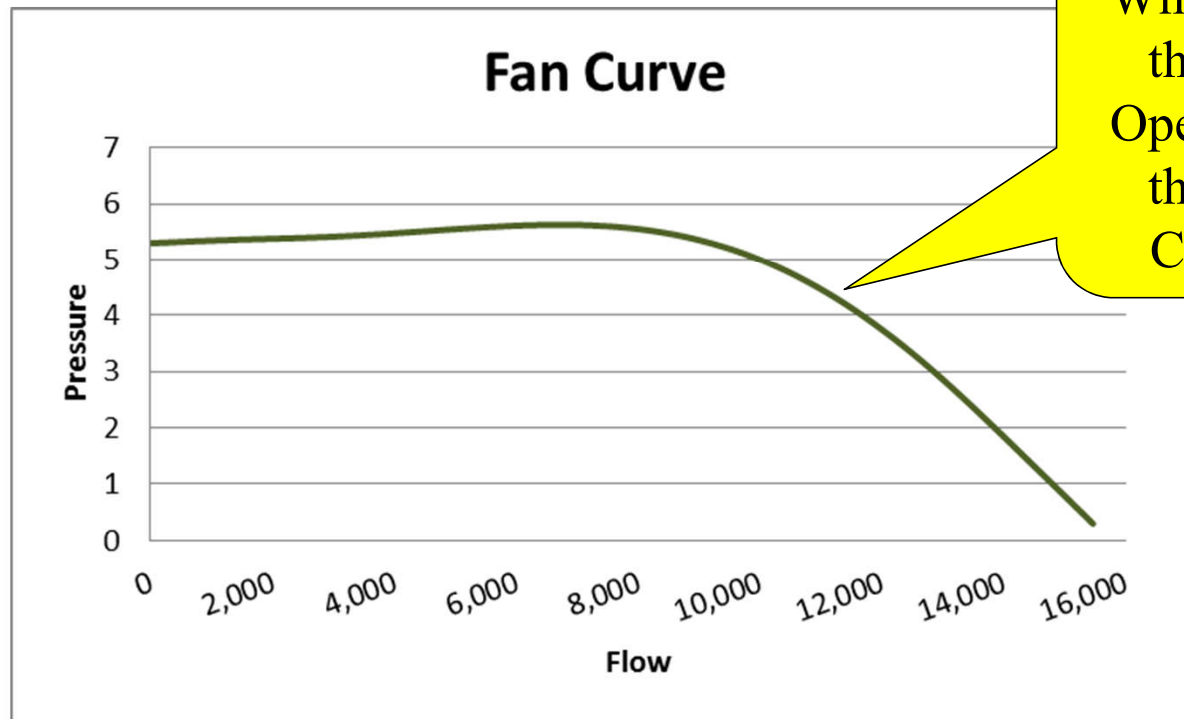
Performance shown is for AF Fans with evase discharges, with outlet ducts, and with or without inlet ducts. BHP does not include belt losses.



Product Selection Software

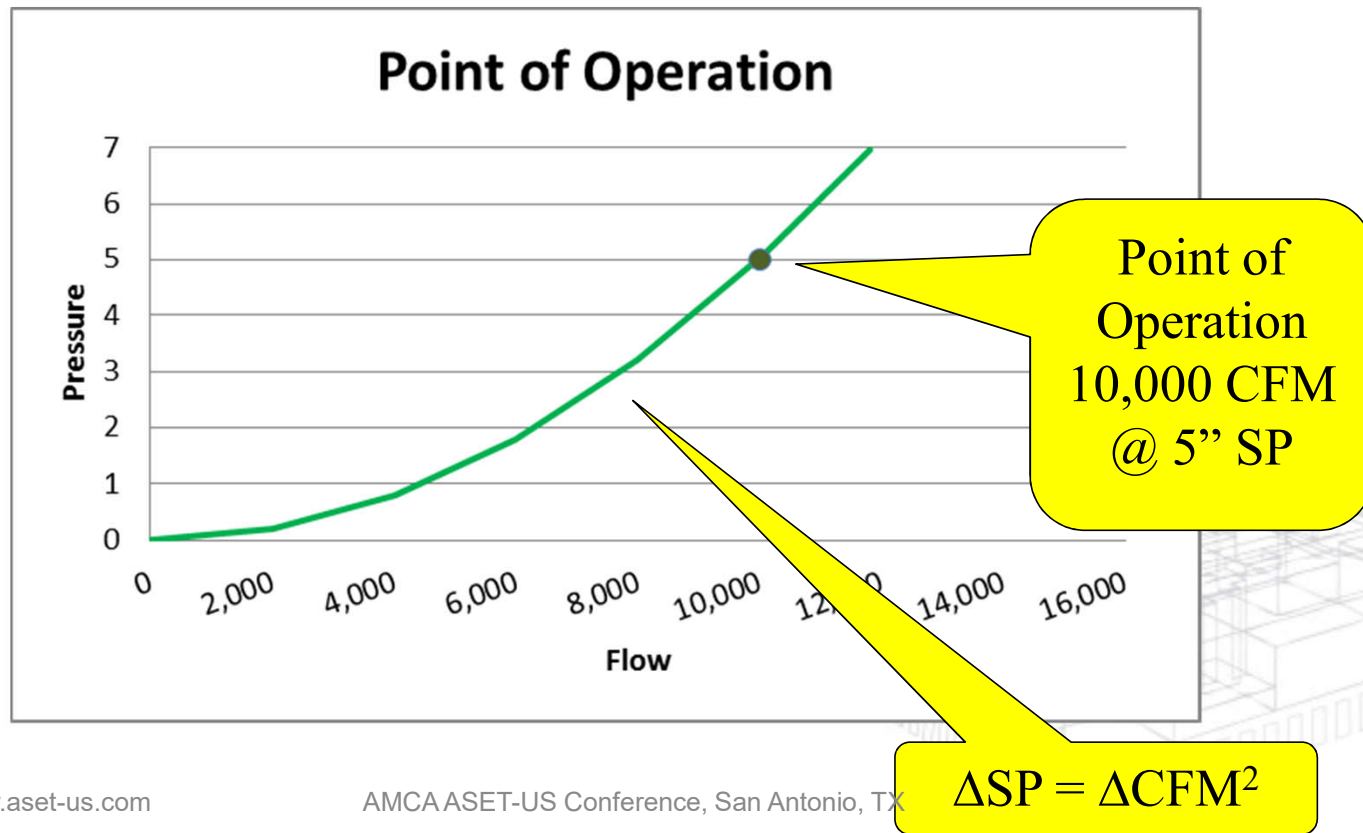


Operation in a System

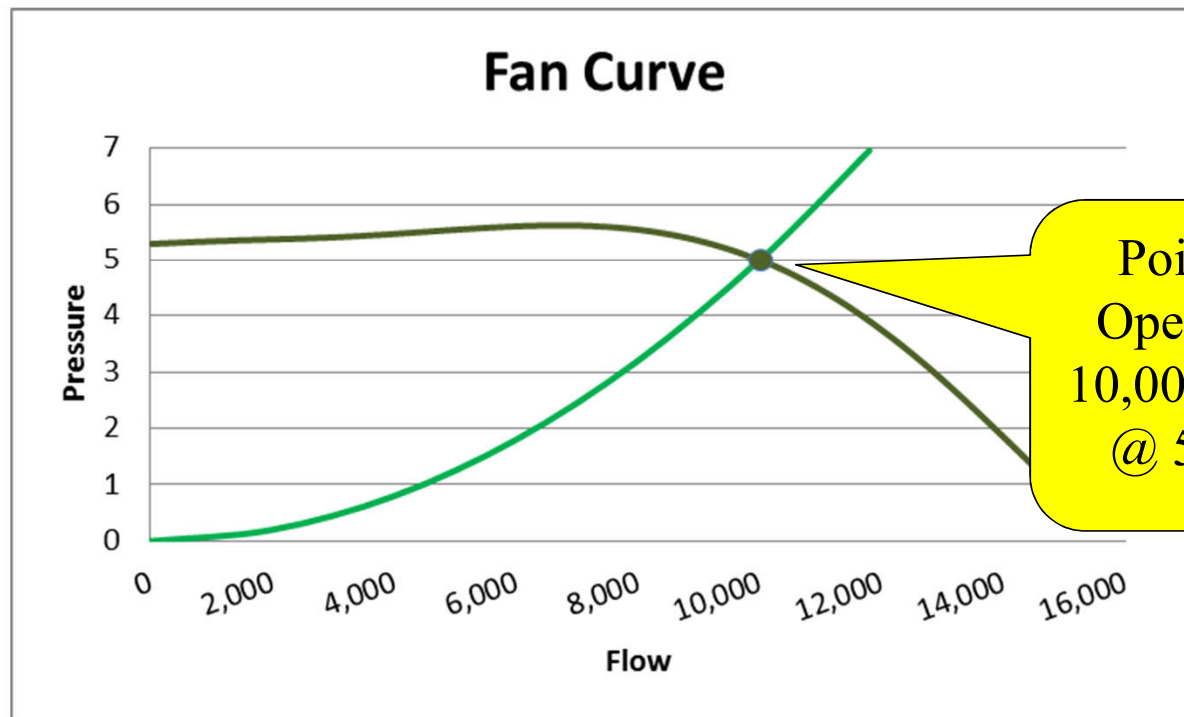


Where will the Fan Operate on the Fan Curve?

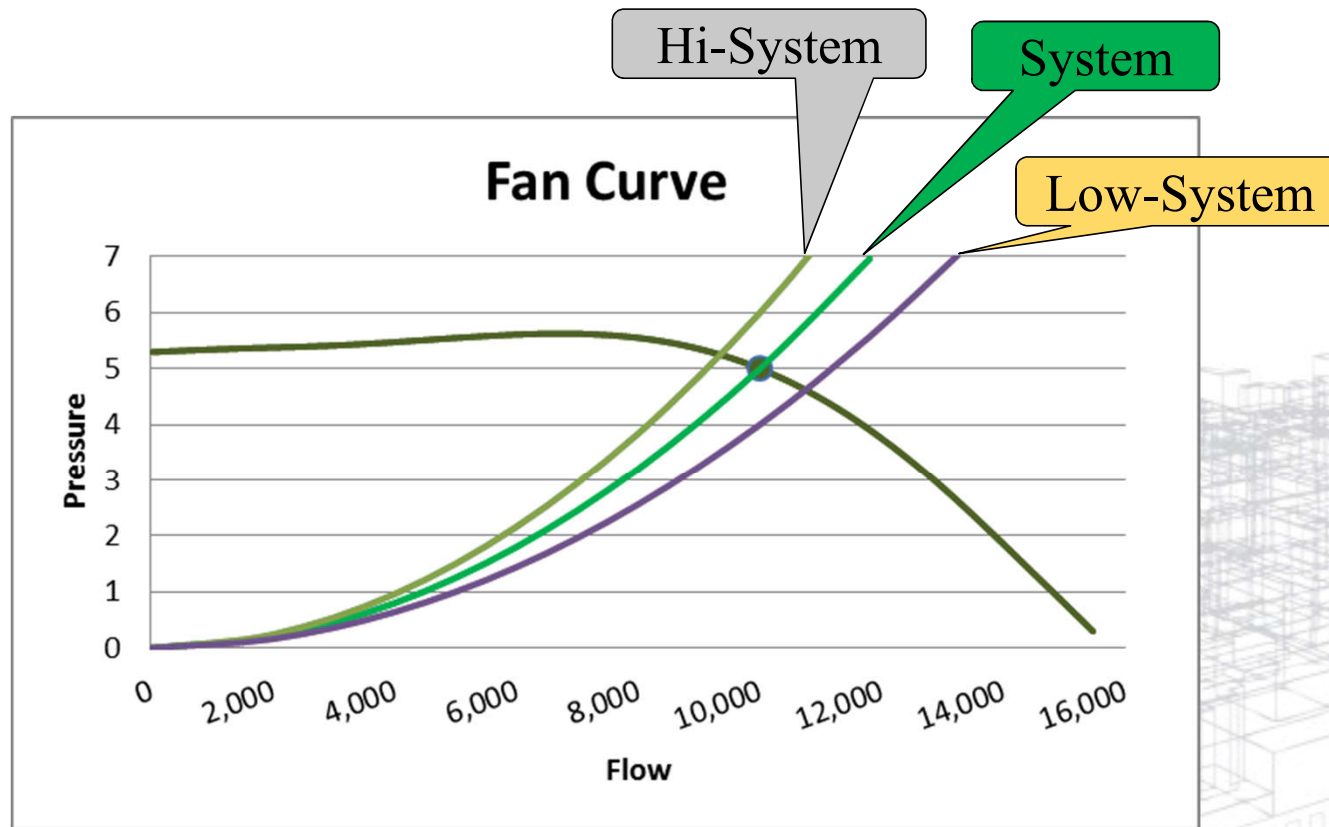
Operation in a System



Operation in a System



Operation in a System

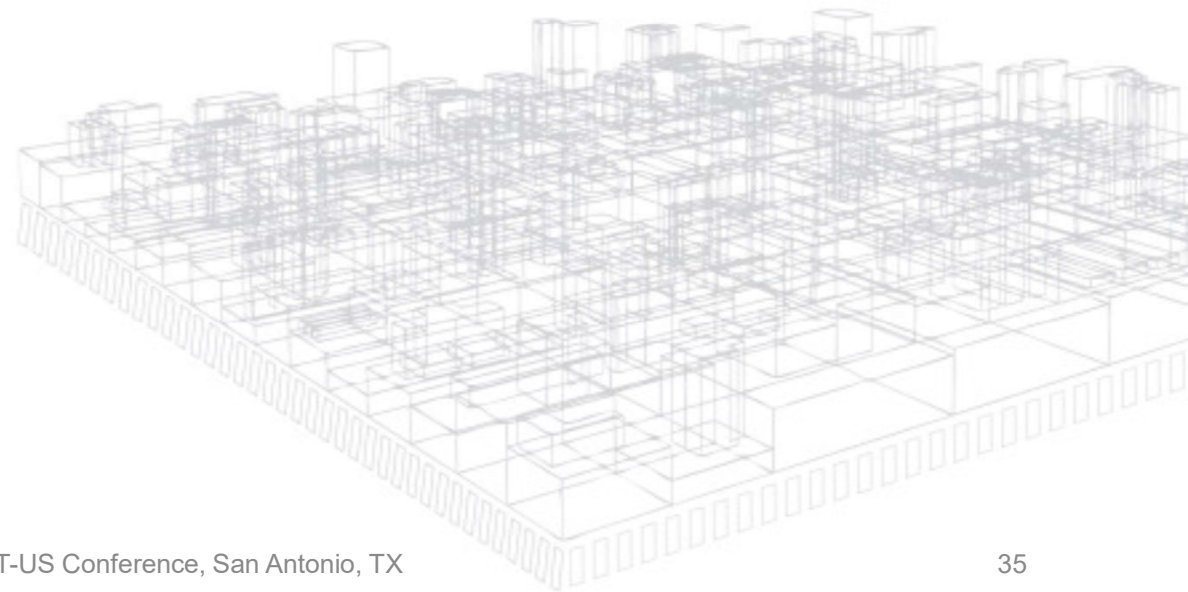


Density and Airstream

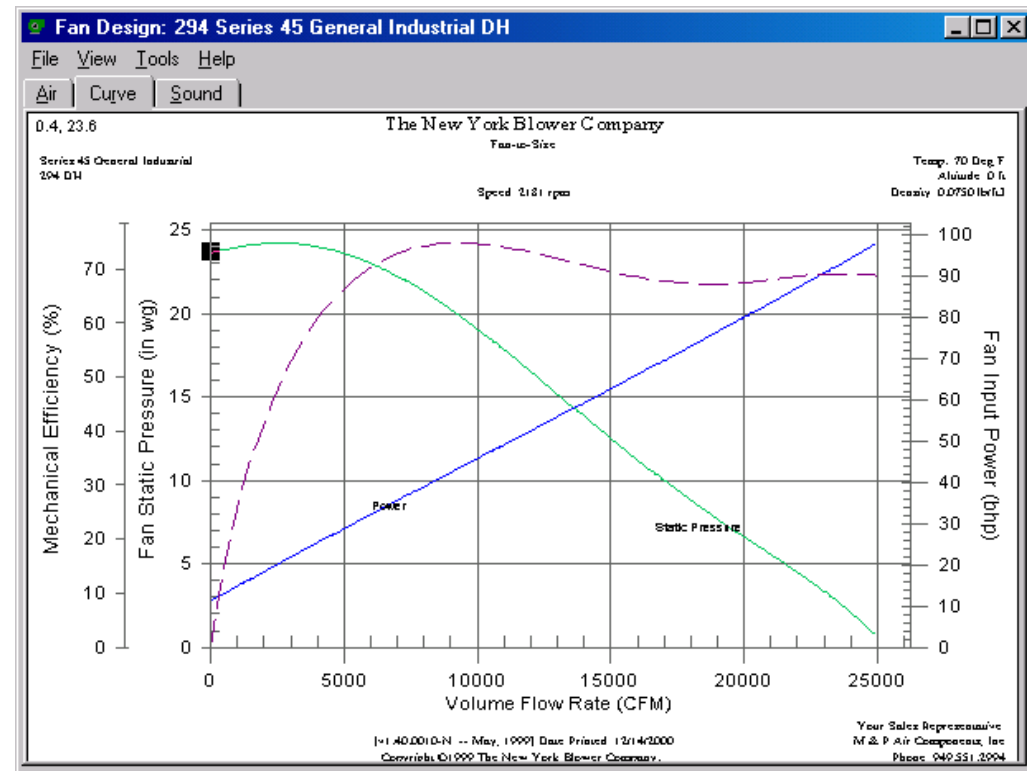
- A fan's operation is impacted by the density and the component make-up of the gas stream moving through the fan.
- The gas stream density is affected by temperature and altitude.
- The primary concern for the operating temperature of a fan is a *mechanical* issue.

Fan Types

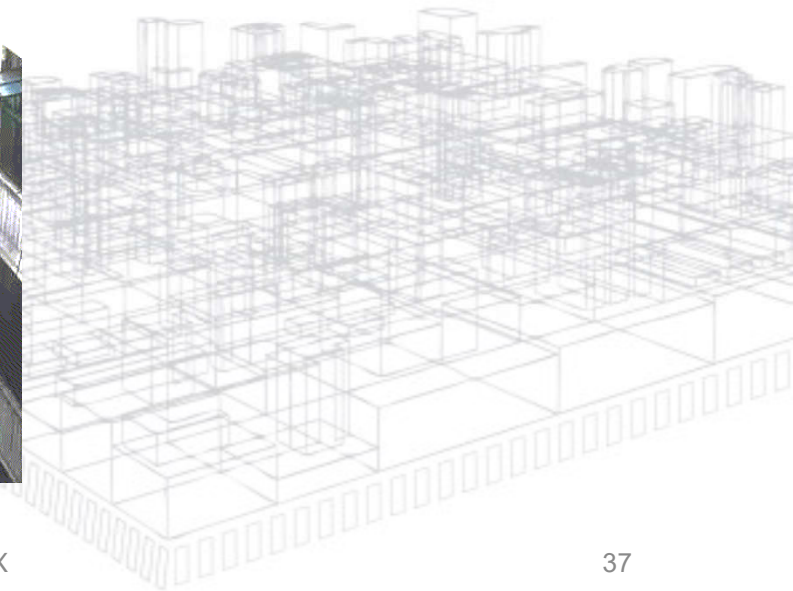
- Different fan (impeller) types have differing characteristic (performance) curves.



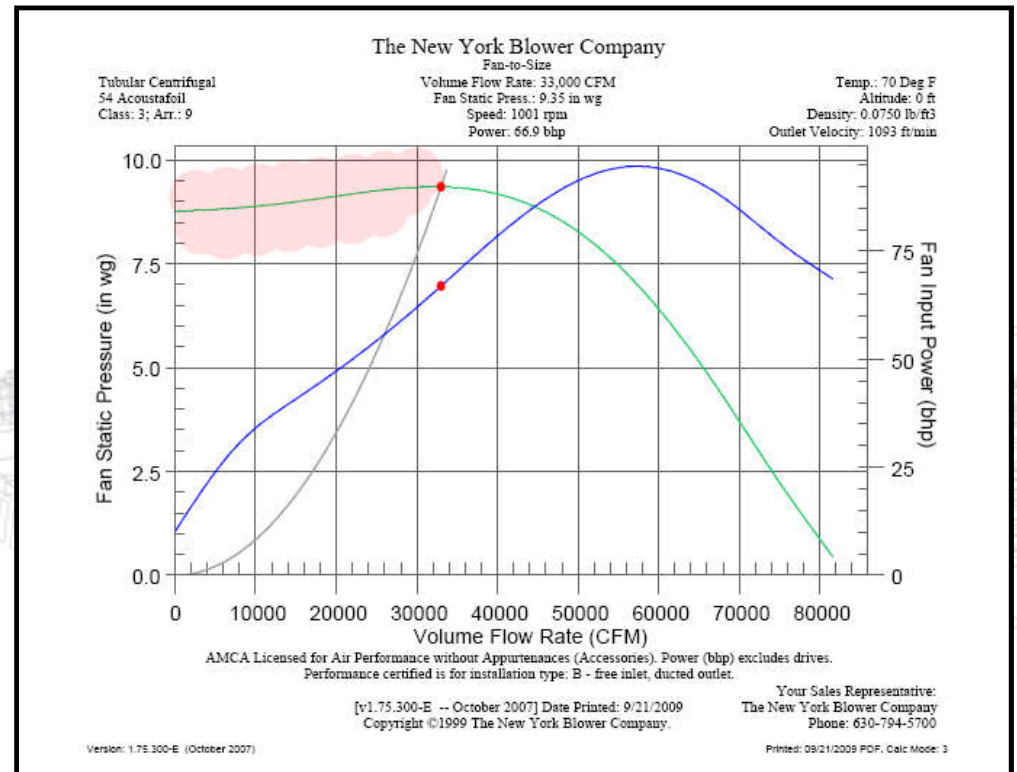
Radial



Radial



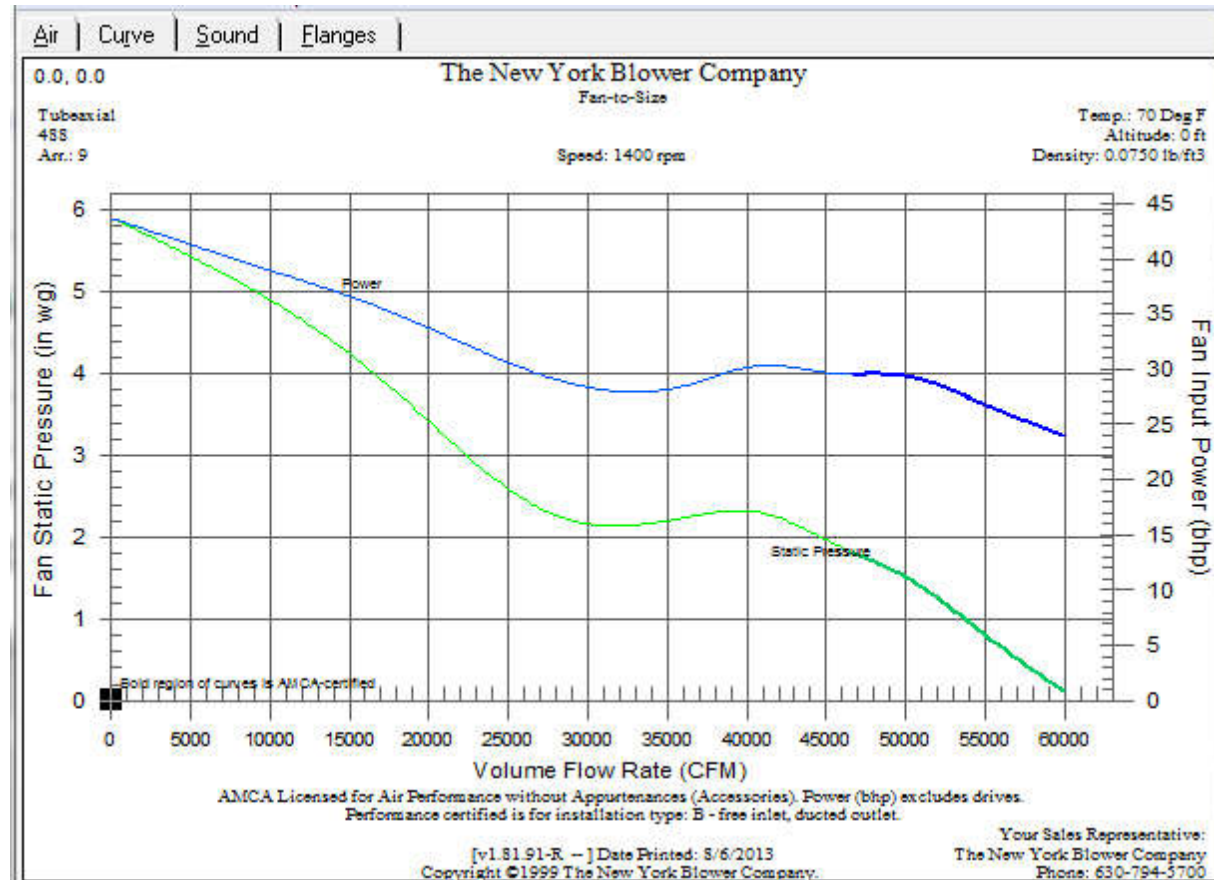
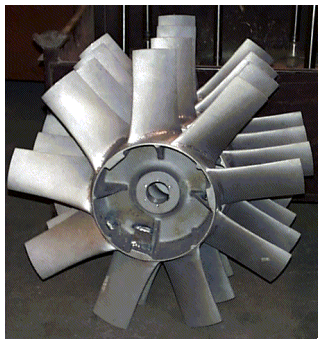
Backward Inclined



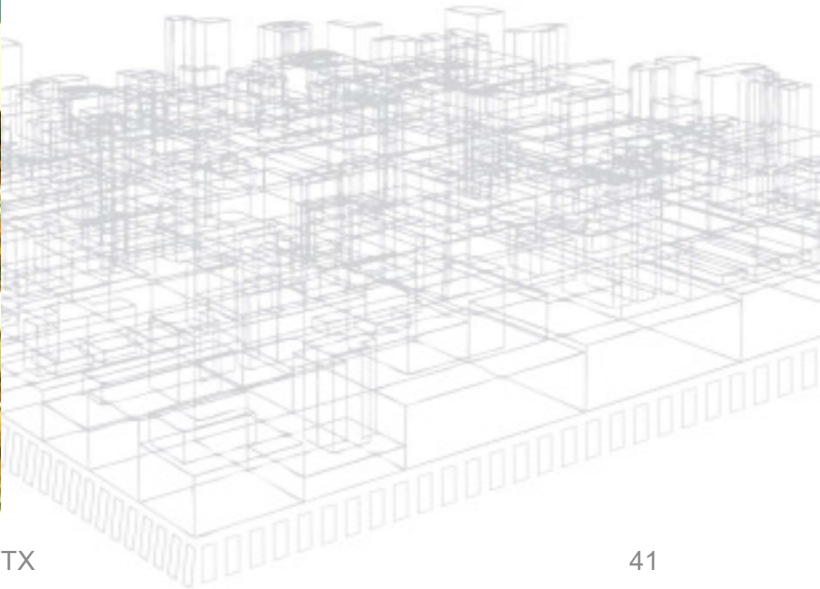
Backward Inclined



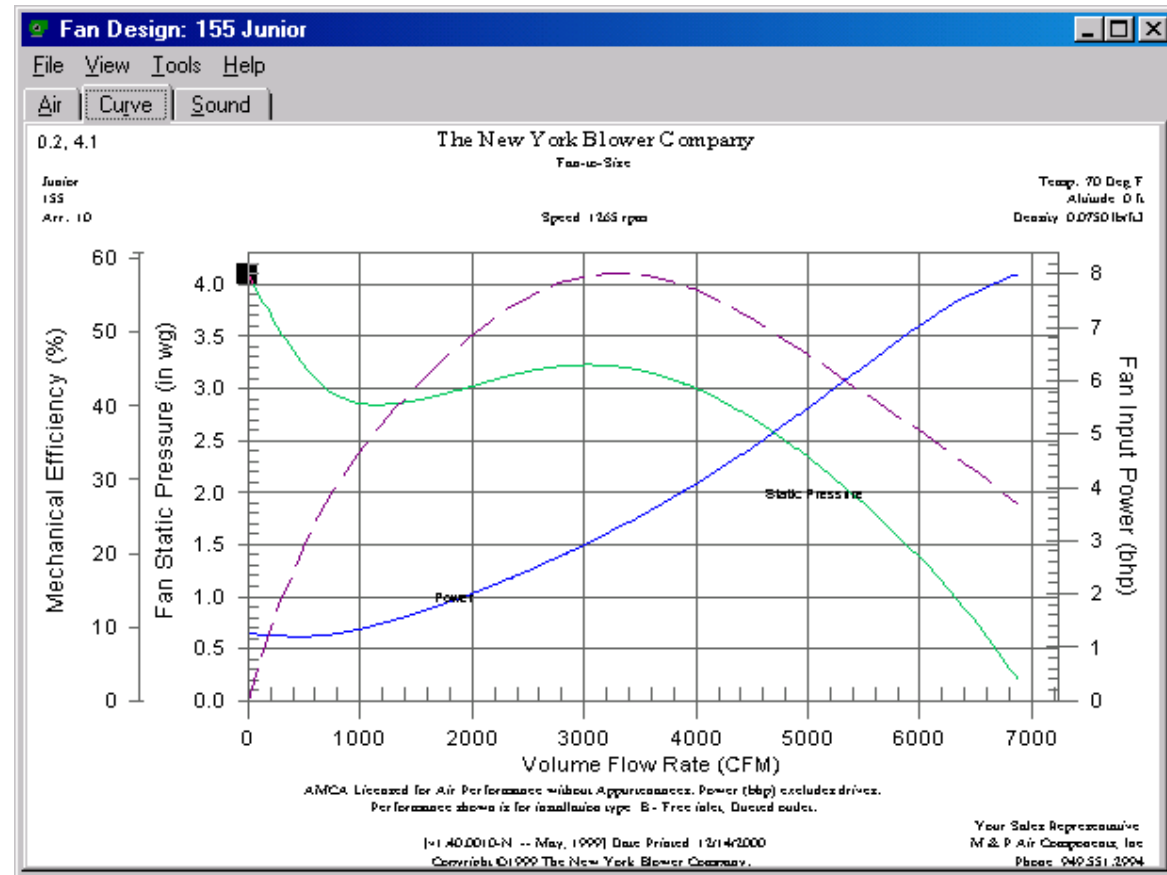
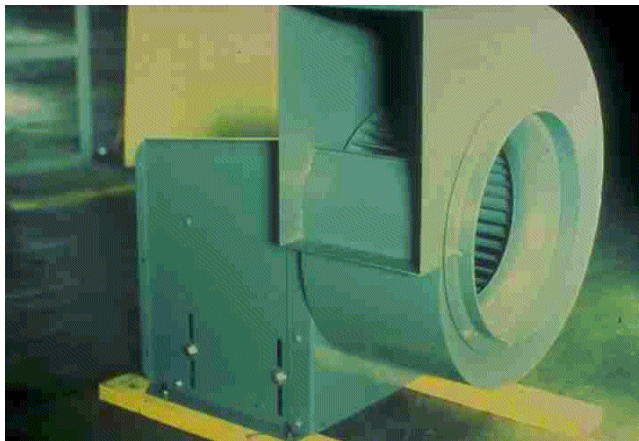
Axial



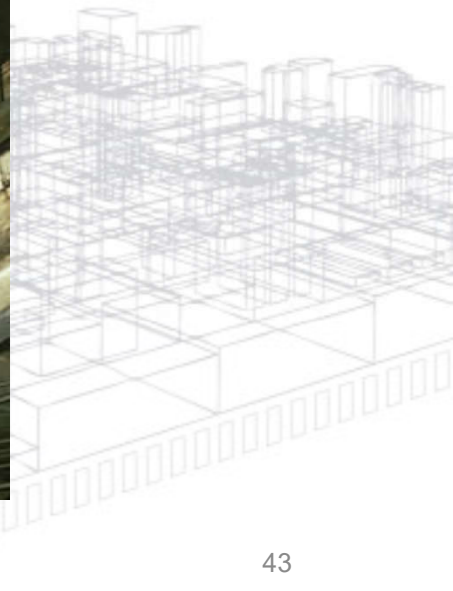
Axial



Forward Curve



Forward Curve

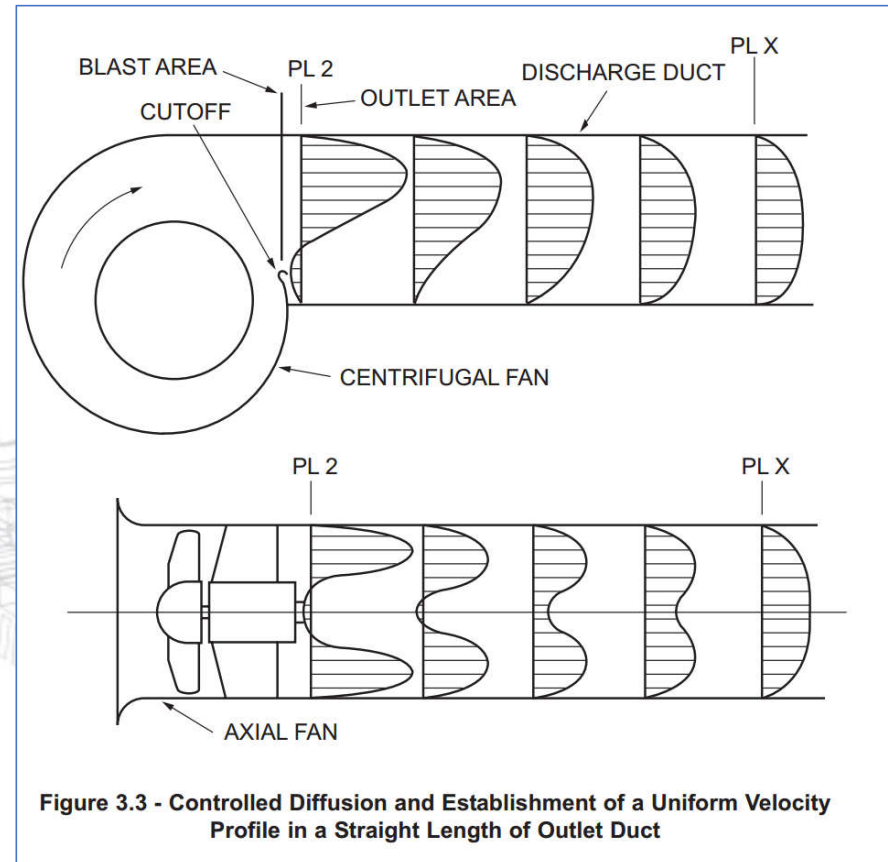
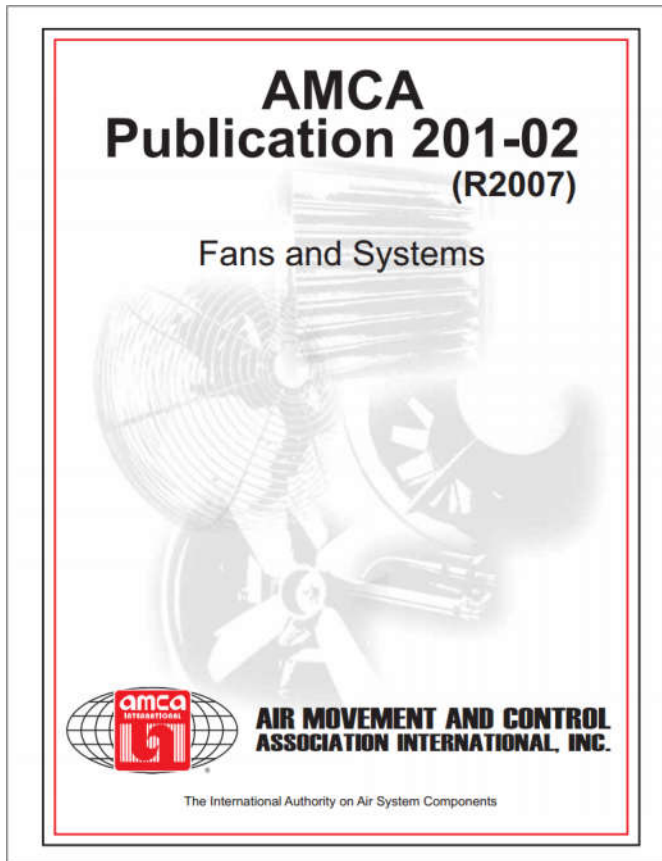


System Effects

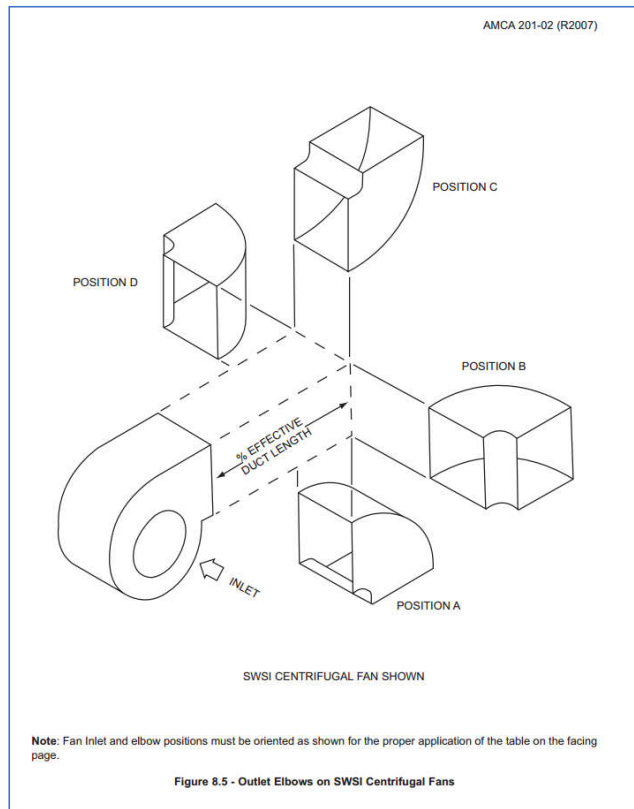
- Laboratory Environment
- Specified (AMCA) Configurations
- Inlet and outlet conditions specified...why?
...to *minimize system effect*



System Effects



System Effects



AMCA 201-02 (R2007)

Blast Area Outlet Area	Outlet Elbow Position	No Outlet Duct	12% Effective Duct	25% Effective Duct	50% Effective Duct	100% Effective Duct
0.4	A	N	O	P-Q	S	NO System Effect Factor
	B	M-N	N	O-P	R-S	
	C	L-M	M	N	Q	
	D	L-M	M	N	Q	
0.5	A	O-P	P-Q	R	T	NO System Effect Factor
	B	N-O	O-P	Q	S-T	
	C	M-N	N	O-P	R-S	
	D	M-N	N	O-P	R-S	
0.6	A	Q	Q-R	S	U	NO System Effect Factor
	B	P	Q	R	T	
	C	N-O	O	Q	S	
	D	N-O	O	Q	S	
0.7	A	R-S	S	T	V	NO System Effect Factor
	B	Q-R	R-S	S-T	U-V	
	C	P	Q	R-S	T	
	D	P	Q	R-S	T	
0.8	A	S	S-T	T-U	W	NO System Effect Factor
	B	R-S	S	T	V	
	C	Q-R	R	S	U-V	
	D	Q-R	R	S	U-V	
0.9	A	T	T-U	U-V	W	NO System Effect Factor
	B	S	S-T	T-U	V	
	C	R	S	S-T	V	
	D	R	S	S-T	V	
1.0	A	T	T-U	U-V	W	NO System Effect Factor
	B	S-T	T	U	V	
	C	R-S	S	T	V	
	D	R-S	S	T	V	

SYSTEM EFFECT CURVES FOR SWSI FANS

DETERMINE SEF BY USING FIGURES 7.1 AND 8.1

For **DWDI** fans determine *SEF* using the curve for SWSI fans. Then, apply the appropriate multiplier from the tabulation below.

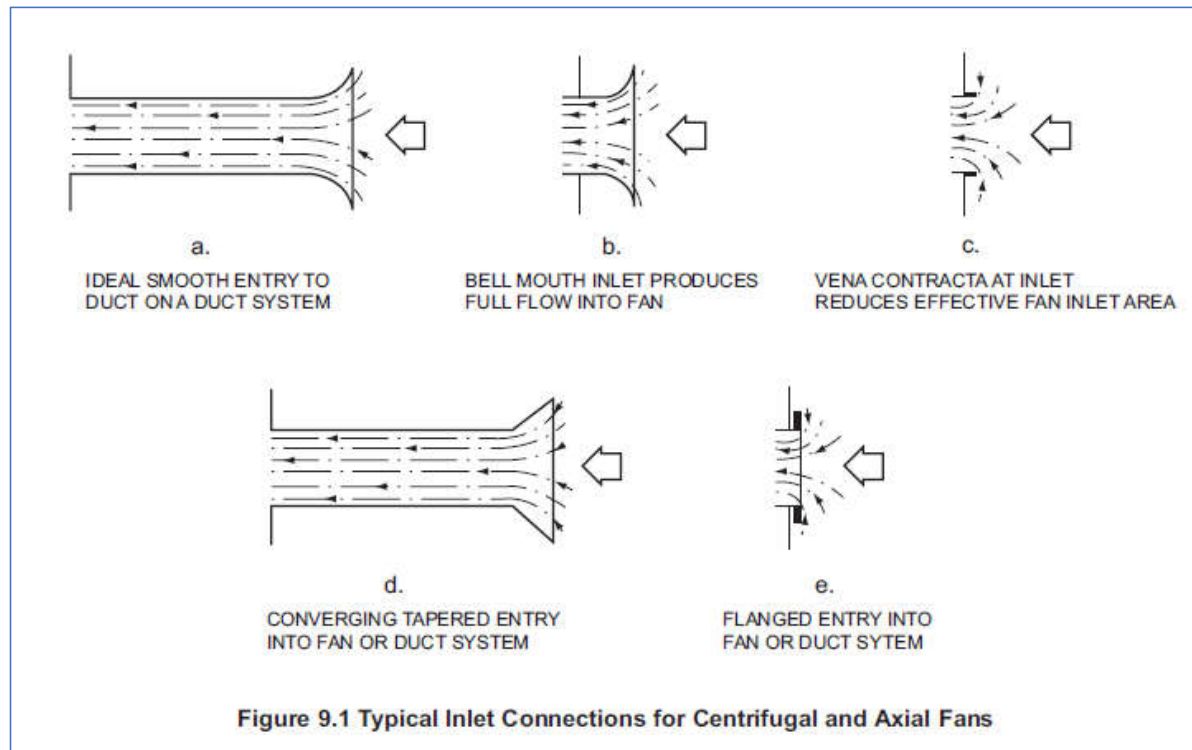
MULTIPLIERS FOR DWDI FANS

ELBOW POSITION A = $\Delta P \times 1.00$
 ELBOW POSITION B = $\Delta P \times 1.25$
 ELBOW POSITION C = $\Delta P \times 1.00$
 ELBOW POSITION D = $\Delta P \times 0.85$

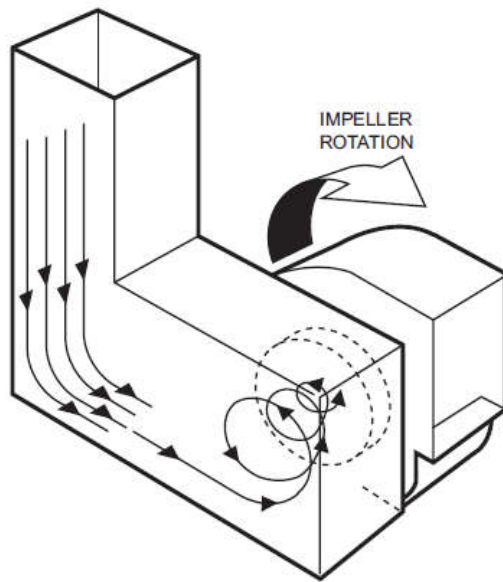
Figure 8.5 - Outlet Elbows on SWSI Centrifugal Fans

34

System Effects



System Effects



COUNTER-ROTATING SWIRL

Figure 9.7 - Example of a Forced Inlet Vortex

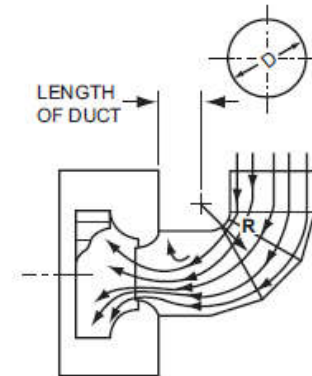


Figure 9.3A - Non-Uniform Airflow Into a Fan Inlet Induced by a 90°, 3-Piece Section Elbow-- No Turning Vanes

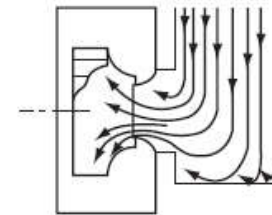
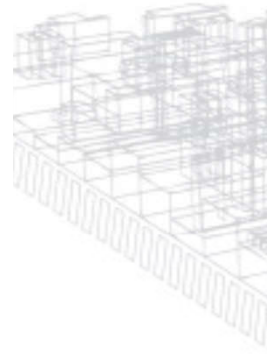
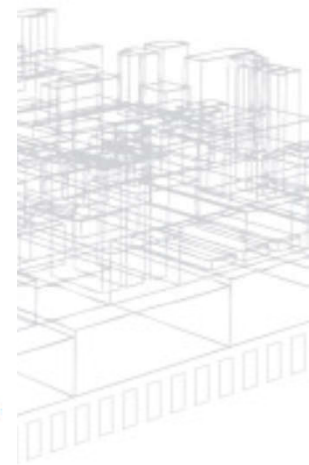


Figure 9.3B - Non-Uniform Airflow Induced Into Fan Inlet by a Rectangular Inlet Duct



System Effects

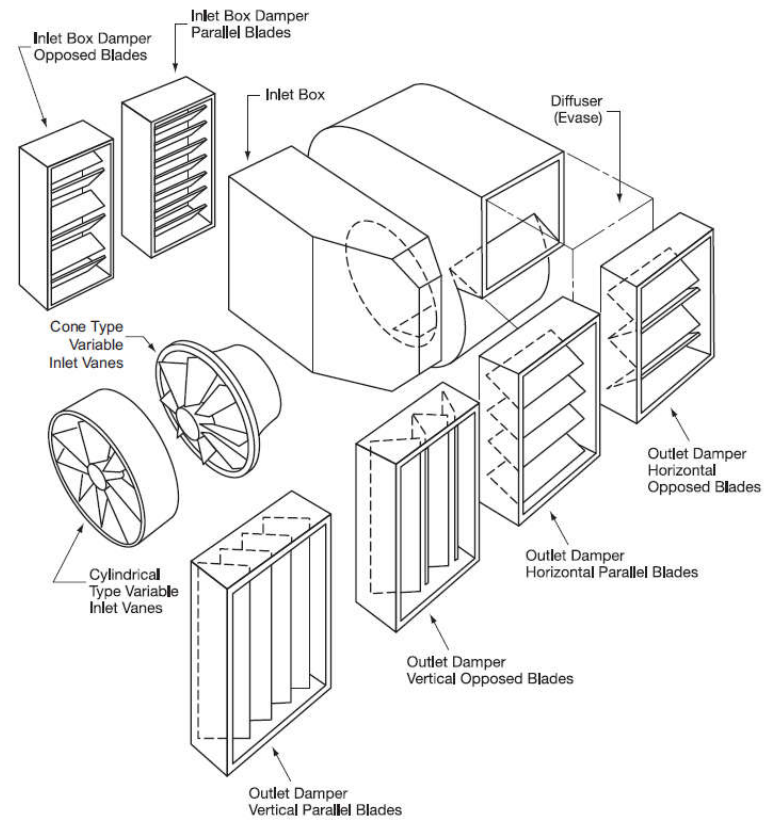
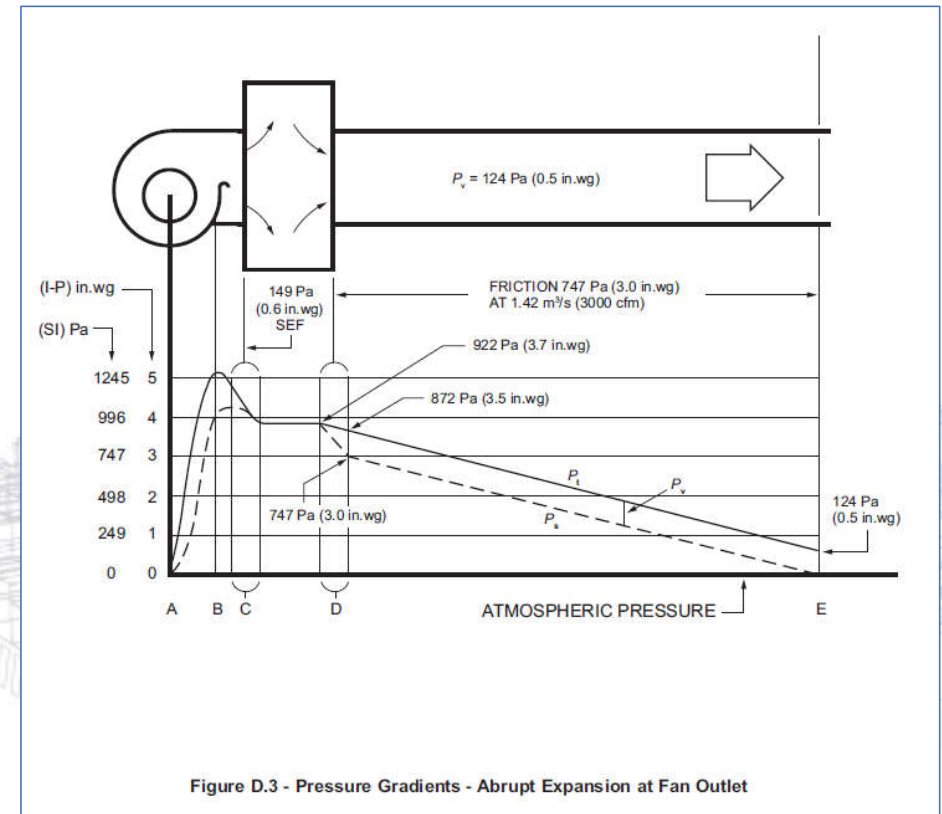
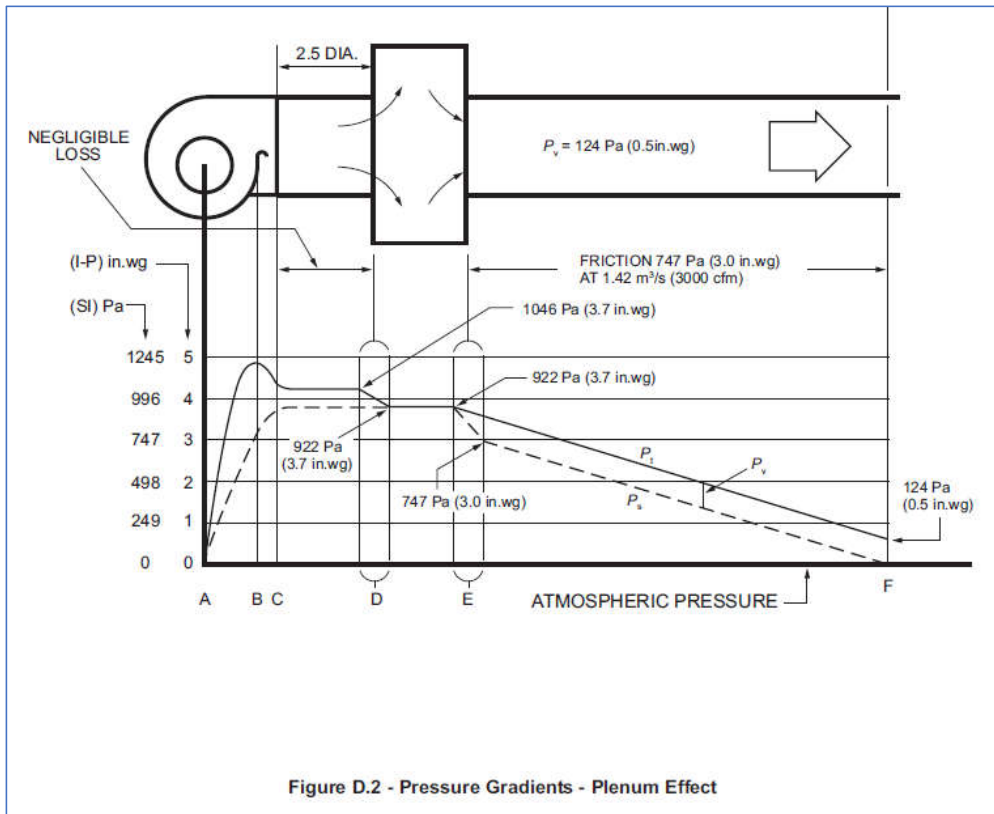
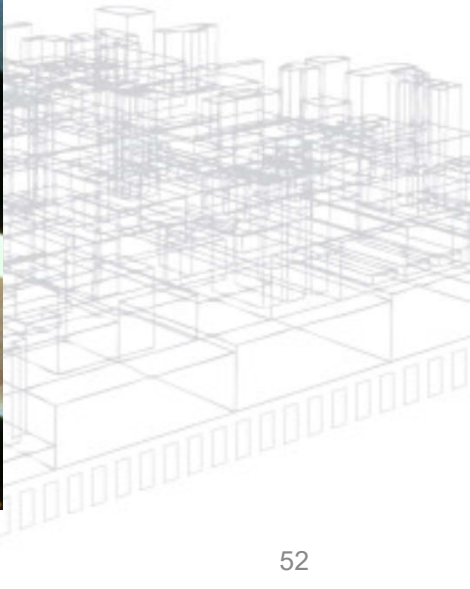
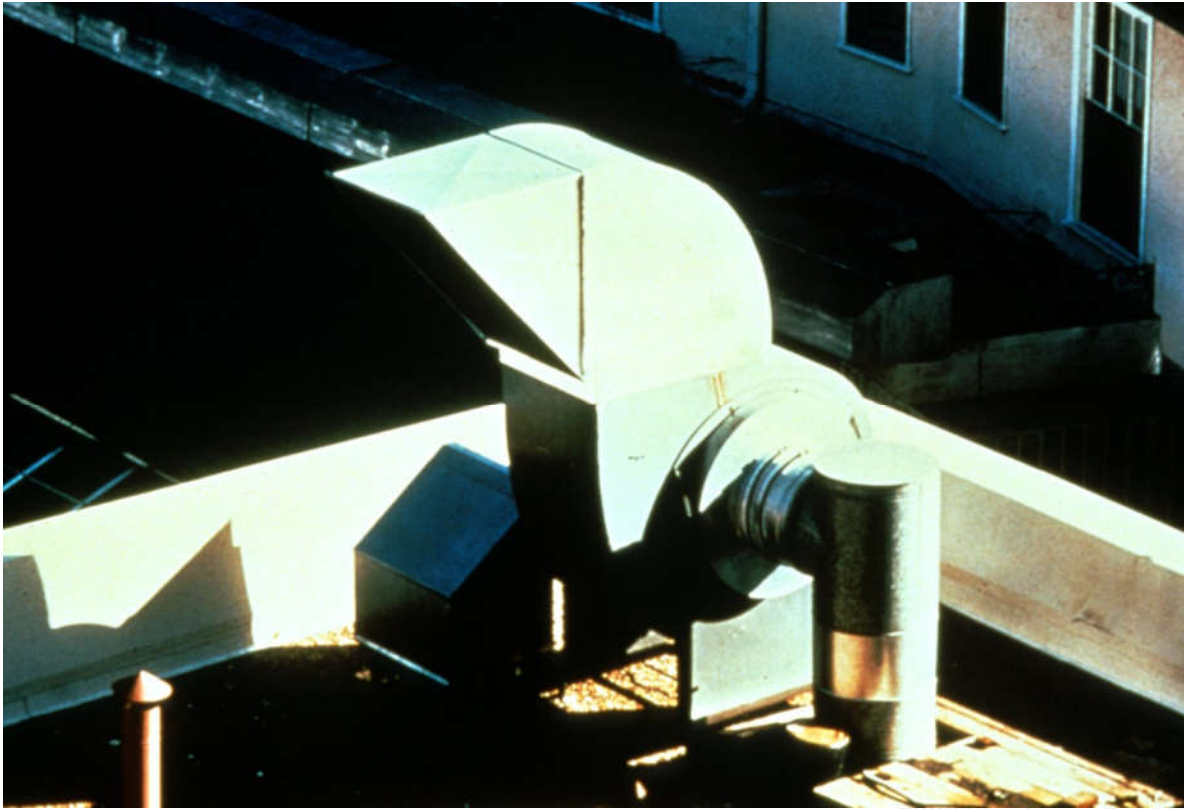


Figure 10.1 - Common Terminology for Centrifugal Fan Appurtenances

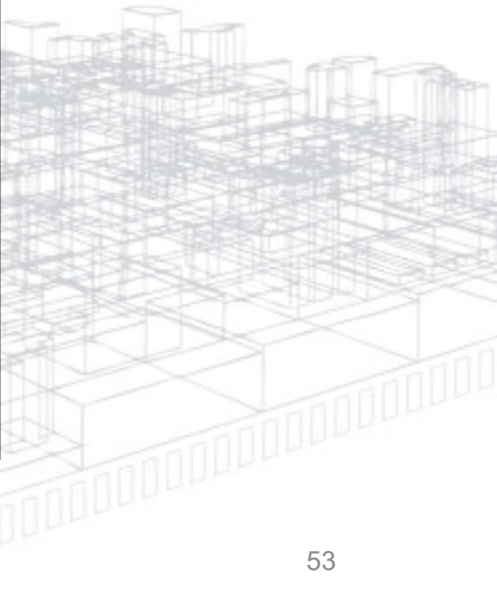
System Effects



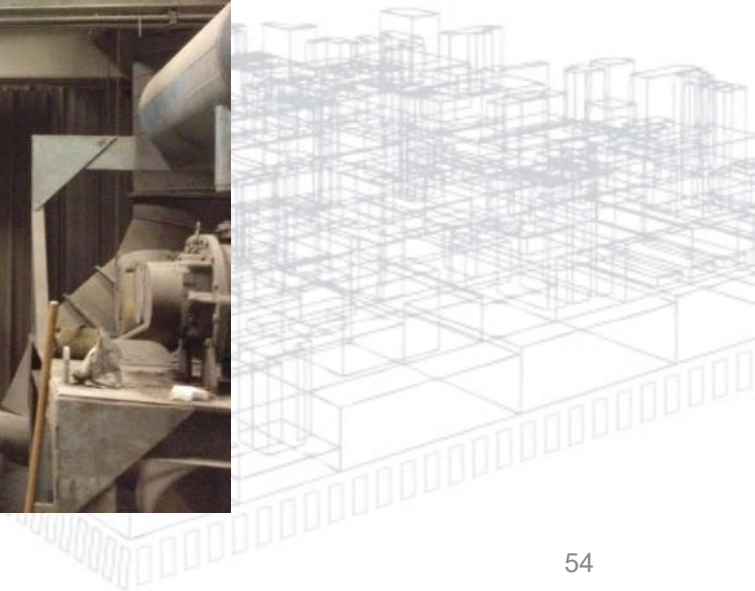
System Effects



System Effects



System Effects



System Effects



March 6–7 2018 www.aset-us.com

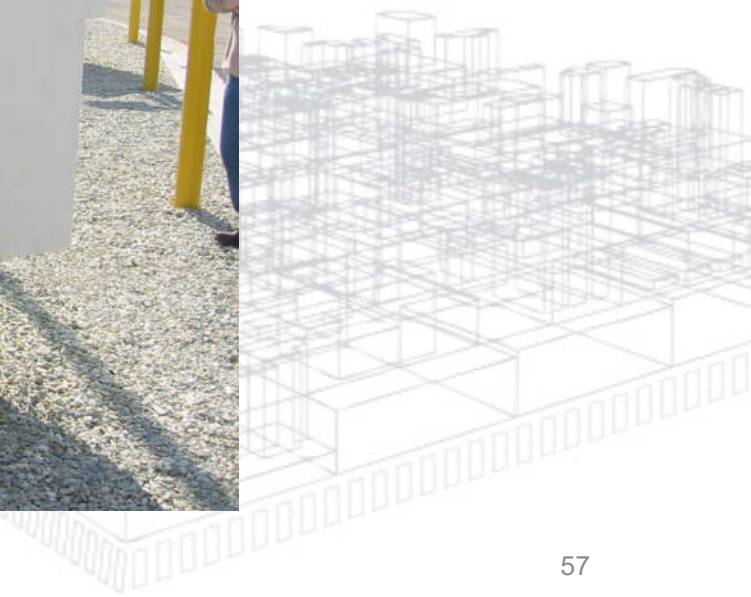
AMCA ASET-US Conference, San Antonio, TX

55

System Effects



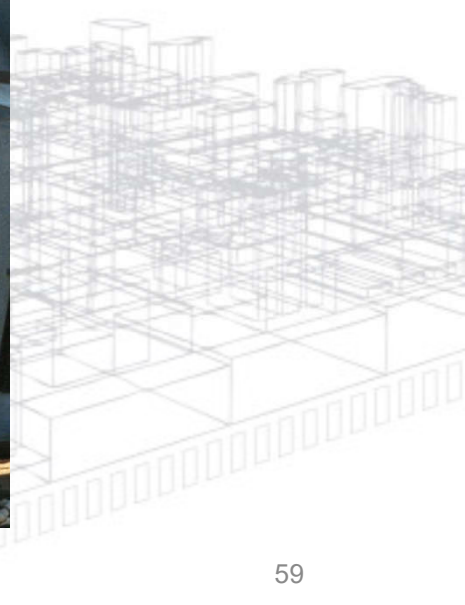
System Effects



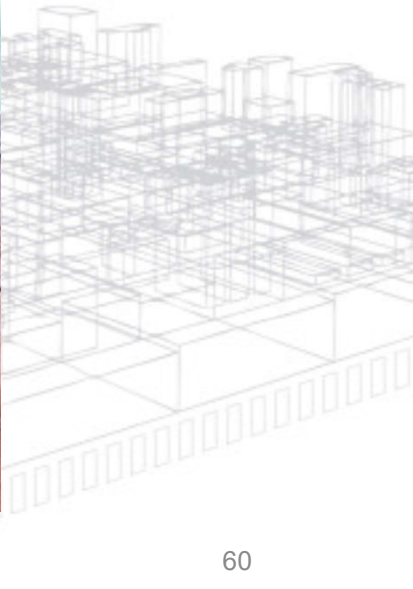
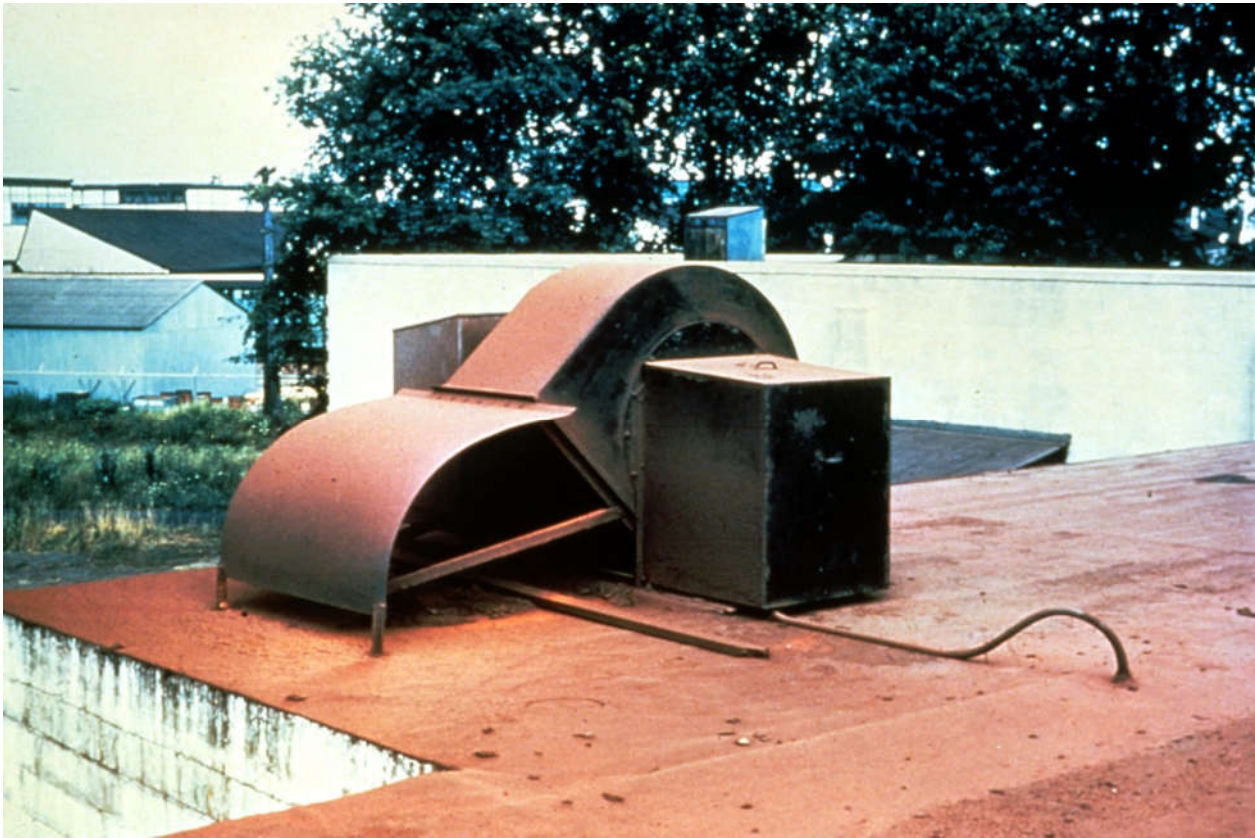
System Effects



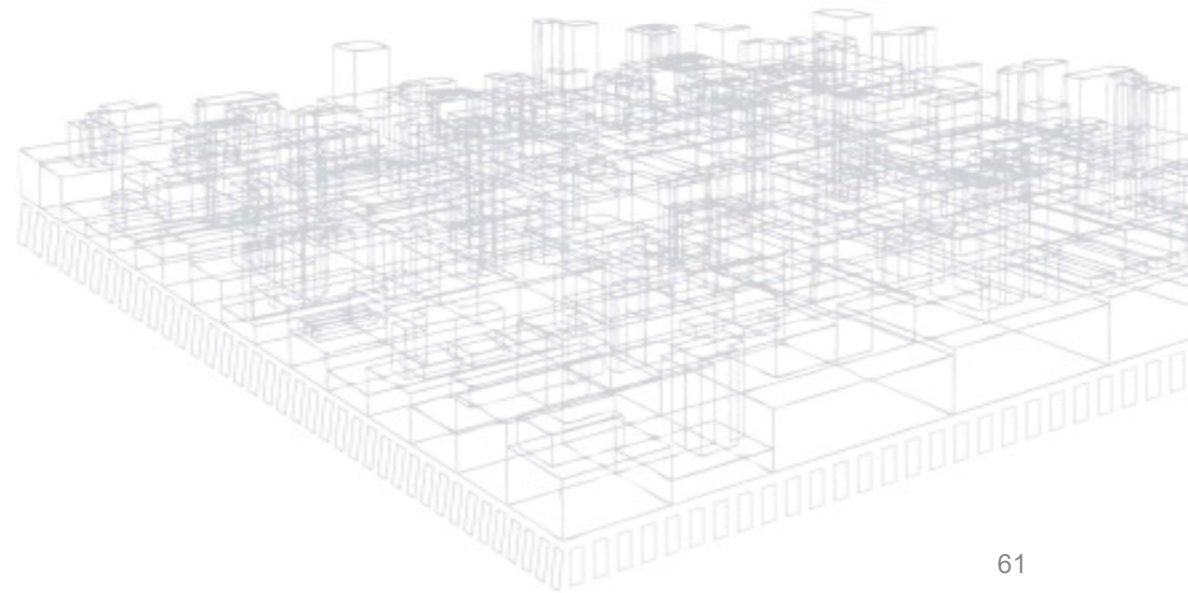
System Effects



System Effects



System Effect Demonstration



Questions?

David Maletich

Director - Marketing

The New York Blower Company

dmaletich@nyb.com

Mark Bublitz

Vice President - Engineering

The New York Blower Company

mbublitz@nyb.com